

Received March 13, 2021, accepted April 7, 2021, date of publication April 9, 2021, date of current version April 20, 2021.

Digital Object Identifier 10.1109/ACCESS.2021.3072332

Intelligent Energy-Based Modified Super Twisting Algorithm and Fractional Order PID Control for Performance Improvement of PMSG Dedicated to Tidal Power System

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This research work was supported by Taif University Researchers Supporting Project number (TURSP-2020/144), Taif University, Taif, Saudi Arabia.

ABSTRACT The majority of marine current conversion technologies are based on permanent magnet synchronous generators (PMSG) due to its numerous advantages such as high-power density, low cost, and favorable electricity production. However, nonlinear properties of the generator and parameter uncertainties, makes the controller design more than a simple challenge. This paper proposes a new adaptive passivity-based (PB) modified super twisting algorithm (PBSTA) for control performance improvement (low tracing errors, fast convergence response, robustness) of a PMSG based marine current energy conversion system under swell effect and parameter uncertainties. The proposed approach combines a new PB current control (PBCC) with a new adaptive modified super twisting algorithm through a fuzzy logic supervisor. A new adaptive fractional order PID (FO-PID) controller is introduced to design the desired dynamics of the system. The main contributions and motivation of this work include the extraction of maximum power from the tidal current, integrating it to the grid and making the closed loop system passive. This is possible by reshaping system energy and introducing a damping term that compensates the nonlinear terms by a damped way and not by cancellation. Two steps are needed to design the proposed controller: the first step includes the derivation of reference current based on the reference torque using adaptive FO-PID. In the second step, the overall control law is computed by the proposed PBSTA. The exponential stability and error convergence of the proposed controller are analytically proven. The developed controller is tested under parameter variations and it is compared to benchmark nonlinear control methods such as sliding mode. Extensive investigation under MATLAB/Simulink, demonstrates clearly that the proposed technique provides higher efficiency and robustness over the benchmark nonlinear control methods.

INDEX TERMS Tidal conversion system, permanent magnet synchronous generator, nonlinear control, passivity-based control.

I. INTRODUCTION

In the electric power sector, the world is seeking to significantly reduce its dependence on fossil fuels, which are

The associate editor coordinating the review of this manuscript and approving it for publication was Qiuye Sun¹.

characterized by high greenhouse gas emissions and unstable prices. At present, offshore tidal turbines are undergoing significant development due to the quality of the resources (depth and speed of the current), availability of abundant power (1.5 MW) and its more predictable behavior compared to wind and solar energies. In this area of research, the grid

connected PMSG based tidal energy generation system is one of the most recent forms of renewable energy conservation problem to be investigated [1], [2].

Recently, various turbines based on permanent magnet synchronous generators (PMSG) have been developed to extract tidal energy. The most widely used technology to convert marine current energy consists of a tidal generator, PMSG, power converters and load. The topology has advantages such as ease of control, low cost, and clean energy. However, the power captured by this tidal conversion system highly depends on the applied control strategies. In fact, nonlinear properties of the generator, parameter uncertainties, and external disturbances, makes the controller design process very complicated. Furthermore, reactive power and DC-link over-voltage supports are the necessary conditions to connect the energy conversion system to the grid [3], [4].

In order to address the aforementioned issues, during the last decades extensive control theories have been investigated. A detailed review on the control strategies developed for PMSG based tidal energy conversion system is presented in [5]. For maximum power harnessing, a sliding mode control (SMC) strategy has been proposed in [6]. However, several factors such as changes in the parameters and sudden variations in the marine current velocity have not been taken into consideration. In [7], a novel active disturbance rejection control method (ADRC) is reported for the PMSG based marine energy conversion system discussed in [6]. The ADRC strategy treats the parameter uncertainties or changes as elements to be rejected which can be canceled during the control design. Compared to SMC and the classical PI control methods, the ADRC method [7] shows clear improvements in the performance of the power conversion system. In [8], a jaya-based sliding mode approach is reported to enhance the control performance of a tidal conversion system. The authors proposed an association of the tidal system with superconducting magnetic energy system (SCMES), for which, the jaya-based controller is applied. However, this association only improves the cost and maintenance time of the conversion system. In [9], a fuzzy sliding mode controller that adaptively extracts the maximum tidal power under swell effects is developed. However, the parameter changes and uncertainties have not been incorporated in the controller design. A magnetic equivalent circuit method-based second-order sliding mode is proposed in [10], however, external disturbances and parameter changes have not been considered in the design process. In order to extract the maximum power from the tidal current under large parametric uncertainties and nonlinearities, a nonlinear observer-based second-order SMC combined with a predictive control was developed in [11]. A linear quadratic controller is proposed in [12]. Using a real profile of the tidal current speed [13], a perturb and observe algorithm is proposed to track the maximum tidal power. Tilt-based fuzzy cascaded control combined with a new Q-network algorithm has been investigated by [14].

Nevertheless, in the aforementioned strategies, the PMSG physical properties are neglected. Thus, this paper

investigates a new passivity-based modified super twisting algorithm (PBSTA) combined with a fractional order PID controller (FO-PID) that maintain the PMSG operating at the optimal torque by tracking its velocity. Inherent advantages of the proposed strategy include the guaranteed robustness properties; the ensured stability and the nonlinear terms are not cancelled but compensated in a damped way. The proposed method makes the system passive, which is possible by introducing a damping term and reshaping its energy to regulate the physical variables to their desired values [15]. The system is decomposed into two interconnected subsystems with negative feedback. The new strategy controls the state dynamics of the electrical part, while the non-electrical dynamics are treated as a “passive disturbance”, unlike the aforementioned nonlinear controls, which usually neglects the PMSG’s mechanical part. To design the desired dynamics, a new adaptive fractional order PID controller via a fuzzy logic method is introduced, which enhances the robustness regardless of the external disturbances and parameter uncertainties of the PMSG. The proposed PBSTA is used to design the controller law, which ensures fast convergences of the measured signals toward their set values and guarantees high stability of the closed-loop system.

In [16], a PBC associated with a sliding-mode control (SMC) is proposed, as mentioned by the authors, the presented combined PBC-SMC control uses more than six fixed gains which is very difficult to determine their optimal values. In fact, as demonstrated by Zhou *et al.* in [7], fixed gains are very difficult to calculate if the control system exhibits parameter variations or uncertainties. In [17], a novel passivity-linear feedback control combined with a fuzzy logic controller is investigated. In [18], a new combined PBC is proposed, two approaches namely, standard passivity-based control and PI-passive controllers are presented. A passivity-based voltage control is developed in [4], however, as mentioned by the authors, the new controller shows a small sensitivity to the variation of the mechanical parameters. A passivity-based control with asymptotic convergence of the states is proposed [19]. The same system was considered in [20]. A PI-PBC is adopted to control the coupling phenomena of the PMSG. In [21], a PBC with a linear feedback controller is investigated for a PMSG based marine energy system, however, parameter uncertainties of the PMSG have not been considered. In [22], a robust adaptive passivity-based control scheme for a class of open-loop unstable nonlinear systems with actuator saturation is proposed. The authors in [23], investigated a passivity-based control combined with a fuzzy control and SMC by constructing a suitable fuzzy function. However, the controller design of the proposed combined strategy is complicated due to the mathematical.

This paper addresses the following two main objectives: controlling the PMSG to guarantee the extraction of maximum tidal power and integrate it to the grid. For this the new hybrid method is presented. The second task is to regulate the reactive power and the DC-link voltage to their references.

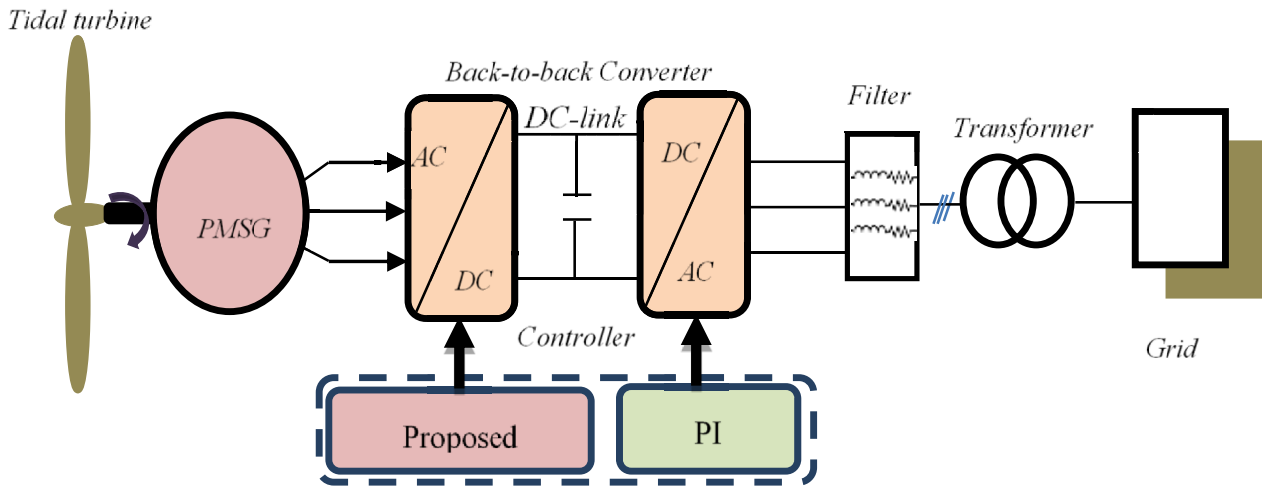


FIGURE 1. Studied system structure.

The disturbances include the nonlinear effects such as PMSG nonlinear friction and swell effect of the tidal velocity. A special attention is given to the generator side converter of the PMSG, as it's the bridge between the tidal turbine and the grid. Furthermore, the robustness against swell effect and parameter uncertainties has been taken into considerations.

The novelty and contribution of the present work are clearly summarized as follows:

- The new passivity-based modified super twisting algorithm combined with adaptive fractional order PID controller is investigated for performance improvement of a PMSG dedicated to marine current conversion system.
- The novel adaptive fractional PID controller via a fuzzy logic method as fuzzy gain supervisor is developed to design the desired dynamics. The fuzzy gain supervisor is used to adaptively adjust gains of the FO-PID which greatly enhance the robustness of the proposed approach against various uncertainties of PMSG.
- When designing the controller, the proposed strategy treats the mechanical characteristics as a passive disturbance, which is compensated in a damped way instead of direct cancellation.
- The global stability of the system and the exponential convergence of the current tracking errors have been analytically proven and further validated by extensive simulation results.

The rest of the paper is organized as follows: System description is established in section 2. Section 3 deals with the formulation of proposed hybrid control strategy. In section 4, GSC controller is discussed. Section 5, presents the numerical validation of the presented control strategy. Finally, the main conclusions are presented in section 6.

II. TIDAL GENERATOR MATHEMATICAL MODEL

The produced power via the generator is controlled by the proposed nonlinear observer-PBVC controller applied to the machine-side converter (see Fig. 1).

A. TIDAL POWER MODEL

The tidal power captured via the turbine and its related output torque, are expressed as follows [4], [7]:

$$P_m = \frac{1}{2} \rho C_p(\beta, \lambda) A v^3 \tag{1}$$

$$T_m = \frac{P_m}{\omega_t} \tag{2}$$

$$C_p(\beta, \lambda) = \frac{1}{2} \left(\frac{116}{\lambda_i} - 0.4\beta - 5 \right) e^{-\left(\frac{21}{\lambda_i}\right)} \tag{3}$$

$$\lambda_i^{-1} = (\lambda + 0.08\beta)^{-1} - 0.035 \left(1 + \beta^3 \right)^{-1} \tag{4}$$

$$\lambda = \frac{\omega_t R}{v} \tag{5}$$

From Eq. 4 and 5, β represents the pitch angle, v represents the tidal speed, λ denotes the tip-speed ratio, R represents the blades radius, ρ denotes the water density, ω_t denote the turbine speed, A represents the area of the blades, and C_p denotes the power coefficient.

B. PMSG MODEL

The adopted dq PMSG model is expressed bellow [4], [15]:

$$v_{dq} = R_{dq} i_{dq} + L_{dq} \dot{i}_{dq} + \psi_{dq} p \omega_m \tag{6}$$

$$J \dot{\omega}_m = T_m - T_e - f_{fv} \omega_m \tag{7}$$

$$T_e = \frac{2}{3} p \psi_{dq}^T i_{dq} \tag{8}$$

where, f_{fv} represents the viscous friction coefficient, $L_{dq} = \begin{bmatrix} L_d & 0 \\ 0 & L_q \end{bmatrix}$ represents dq inductances matrix, J is the moment of inertia, $i_{dq} = \begin{bmatrix} i_d \\ i_q \end{bmatrix}$ represents the stator current vector,

T_e represents the electromagnetic torque, $\psi_{dq} = \begin{bmatrix} \psi_f \\ 0 \end{bmatrix}$ represents the flux linkages vector, $v_{dq} = \begin{bmatrix} v_d \\ v_q \end{bmatrix}$ represents

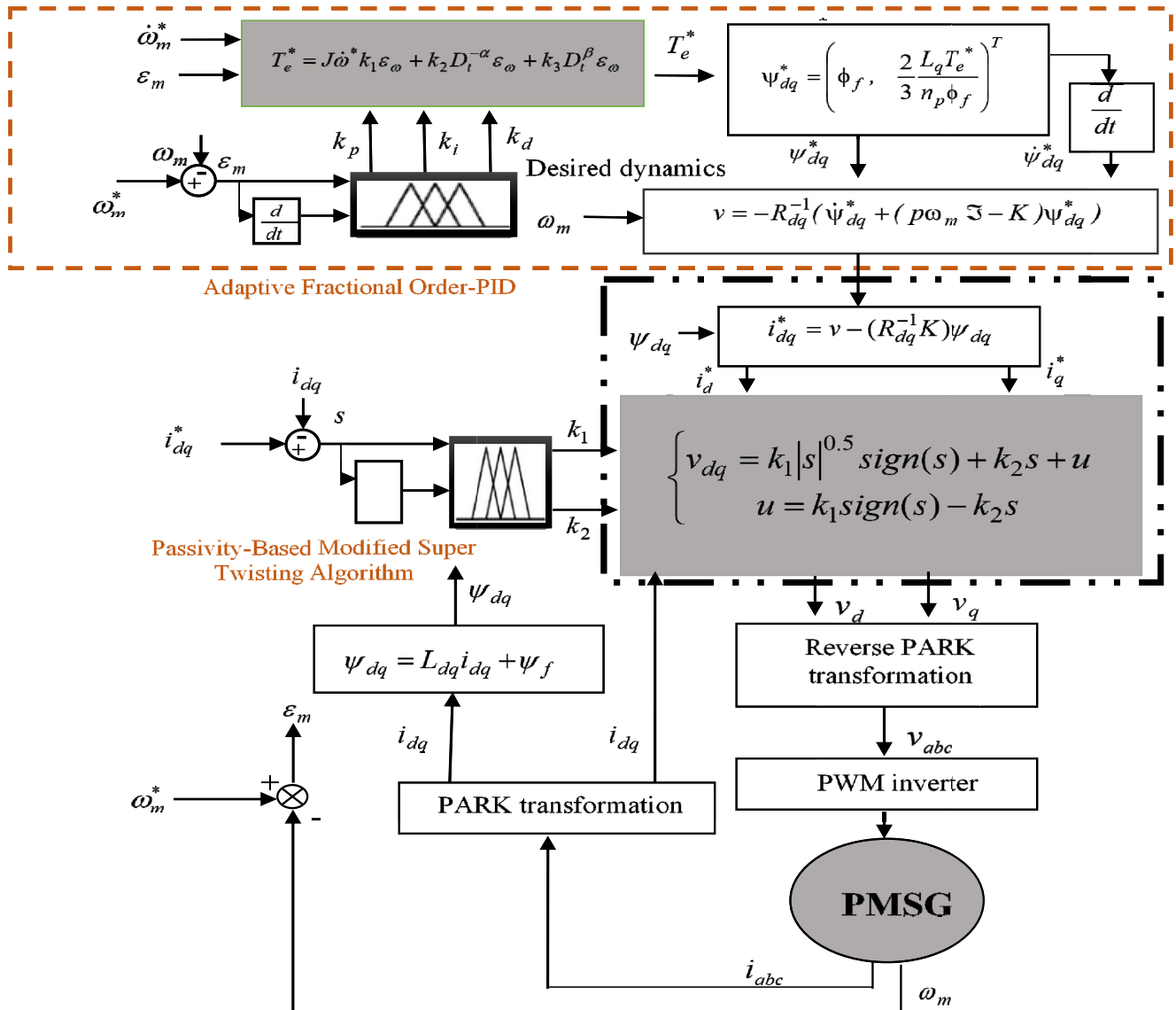


FIGURE 2. Proposed strategy design.

voltage stator vector, $R_{dq} = \begin{bmatrix} R_s & 0 \\ 0 & R_s \end{bmatrix}$ represents the stator resistance matrix.

III. PROPOSED CONTROLLER DESIGN PROCESS

The design of the new control strategy requires a number of steps: First, it is necessary to calculate an Euler-Lagrange model, to choose an appropriate input and output vectors such that the relationship between them is passive. Second, the system has to be decomposed into two interconnected subsystems with negative feedback. Finally, the last step consists to identify the non-dissipative terms in the system model. The controller design process is depicted in Fig. 2. Main idea for introducing the PBC control is to make the dynamics of the closed loop system passive and this is achieved by introducing a damping term and reshaping system's energy which

compensates the nonlinear phenomena's in a damped way instead of direct cancellation. The controller design process is depicted in Fig. 2, in which, two main parts can be distinguished: The first step, is the computation of the reference electromagnetic torque, computed by the new adaptive FO-PID, and then from the reference torque, the desired current is obtained. The second part computes the control voltage using the proposed PBSTA controller.

A. PASSIVITY-BASED MODIFIED SUPER TWISTING ALGORITHM DESIGN

In [16], a proportional-integral (PI) control is proposed for the dq axis currents and to track its set vector $i_{dq}^* = \begin{bmatrix} i_d^* \\ i_q^* \end{bmatrix}$. However, as demonstrated by Zhou et al. in [7], fixed gains are very difficult to calculate if the control system exhibits

parameter variations or uncertainties. Thus, the PBSTA controller is proposed to improve the robustness and resolve the problemes faced by the PI loops. Thus, the controller output voltages (v_{dq}) of the PMSG is computed as follows:

$$\begin{cases} v_{dq} = k_1 |s|^{0.5} \text{sign}(s) + k_2 s + u \\ u = k_1 \text{sign}(s) - k_2 s \end{cases} \quad (9)$$

$$s = (i_{dq}^* - i_{dq}) \quad (10)$$

where, $k_1 > 0$ and $k_2 > 0$. $s = (i_{dq}^* - i_{dq})$, represent the sliding surface. The proposal is to find v_{dq} which ensures the convergence of the system dynamic. As indicated previously, fixed gains are very difficult to calculate if the system exhibits parameter variations. To overcome this problem, a fuzzy logic controller is used as gain supervisor. Then, the controller output v_{dq} is designed as shown in Fig. 3.

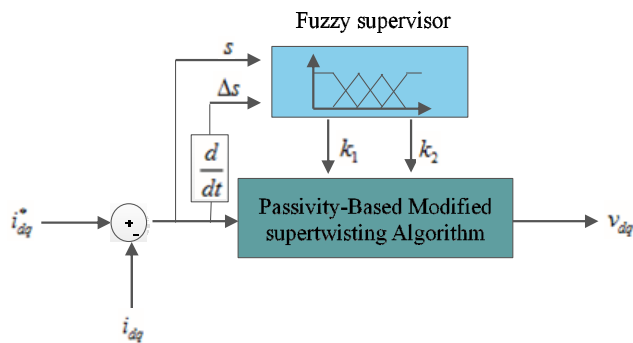


FIGURE 3. Controller law with the proposed adaptive PBSTA.

The fuzzy supervisor is used for the adaptation of the gains k_1 and k_2 of the PBSTA and thus, it solves the problem caused by imprecise parameters. The fuzzy inputs are chosen as the current error s and its derivative. The fuzzy control consists of three steps: In the first step the inputs are fuzzified, rules are formulated as a second step and finally the output is defuzzified. The types of the membership functions used are triangular and trapezoidal type uniformly distributed and symmetrical in the universe of discourse (see Fig. 4). The method of partitioning these functions is given according to Lee and Takagi [24] and Yubazaki *et al.* [25]. Their method is based on the idea of sharing the same parameter by several membership functions. The advantage of this method is that the number of parameters of the membership functions is significantly reduced. In Table 1, the linguistic variables corresponding to the inputs-outputs of the fuzzy gain scheduling are chosen as: Negative Big (NB), Negative Small (NS), Zero (Z), Positive Big (PB), and Positive Small (PS). The decision-making output is obtained using a Max-Min fuzzy inference where the crisp outputs are calculated by the center of gravity defuzzification method [26].

Remark 1: The choice of the fuzzy logic system to compute the gains of the modified super twisting algorithm is justified by the fact that fixed gains are complicated to calculate when the system is exposed to parameter uncertainties [26]. The

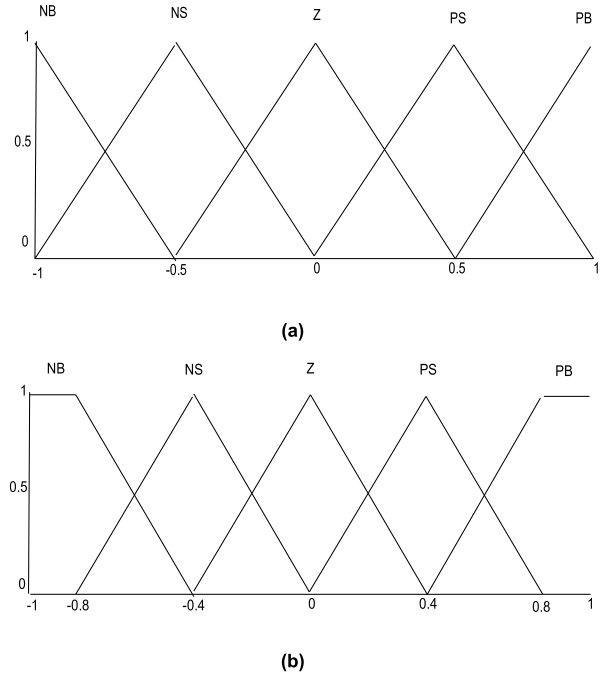


FIGURE 4. The fuzzy controller configuration. (a) Input membership function, (b) Output membership function.

TABLE 1. Fuzzy logic rules.

$\Delta \varepsilon_\omega$ \ ε_ω	NB	NS	Z	PS	PB
NB	NB	NB	NS	NS	Z
NS	NB	NB	NS	Z	PS
Z	NS	NS	Z	PS	PS
PS	NS	Z	PS	PB	PB
PB	Z	PS	PS	PB	PB

reader is referred to [27] for more information about the super twisting control and its advantages.

B. PMSG PASSIVE FEEDBACK DECOMPOSITION

The PMSG input-output relationship for electrical part (6) can be formulated by the following expression:

$$\Sigma_e : V_e = \begin{bmatrix} v_{dq} \\ -\omega_m \end{bmatrix} \rightarrow Y_e = \begin{bmatrix} i_{dq} \\ T_m \end{bmatrix} \quad (11)$$

The mechanical part given in relation (7) can be expressed as follows:

$$\Sigma_m : V_m = (-T_e + T_m) \rightarrow Y_e = \frac{(-T_e + T_m)}{Js + f_{fv}} \quad (12)$$

Thus, the following lemma given below is introduced:

Lemma 1: According to the aforementioned conditions, the PMSG in the dq-model is divided into feedback interconnected two subsystems, i.e. electrical subsystem Σ_e and mechanical subsystem Σ_m .

Proof: From (11), total energy term H_e is given as follows:

$$H_e = \frac{1}{2} i_{dq}^T L_{dq} i_{dq} + \psi_{dq}^T i_{dq} \quad (13)$$

The time derivative of H_e along (6), yields the following expression:

$$\dot{H}_e = -i_{dq}^T R_{dq} i_{dq} + Y_e^T V_e + \frac{d}{dt} (\psi_{dq}^T i_{dq}) \quad (14)$$

Integrating both sides of (12) along $[0 T_e]$, yields:

$$\underbrace{H_e(T_e) - H_e(0)}_{\text{Stored Energy}} = - \underbrace{\int_0^{T_e} i_{dq}^T R_{dq} i_{dq} d\tau}_{\text{Dissipated Energy}} + \underbrace{\int_0^{T_e} Y_e^T V_e d\tau + [\psi_{dq}^T i_{dq}]_0^{T_e}}_{\text{Supplied Energy}} \quad (15)$$

where $H_e(T_e) \geq 0$ and $H_e(0)$ represents the initial energy stored. By integrating Eq. (15), a dissipation inequality is deduced, given as follows:

$$\int_0^{T_e} Y_e^T V_e d\tau \geq \lambda_{\min} \{R_{dq}\} \int_0^{T_e} \|i_{dq}\|^2 - \left(H_e(0) + [\psi_{dq}^T i_{dq}]_0^{T_e} \right) \quad (16)$$

where, $\|\cdot\|$ represents the vector norm of the standard Euclidian.

From Eq. (16), it is established that Σ_e is passive. Then, from Σ_m , the transfer function $F_m(s)$ is expressed by the following expression:

$$F_m(s) = \frac{Y_m(s)}{V_m(s)} = \frac{1}{Js + f_{jv}} \quad (17)$$

It can be deduced that Σ_m is passive, since $F_m(s)$ is a strictly positive function. Thus, the PMSG model is decomposable into two passive sub systems which mean that the PMSG is passive.

C. DESIRED CURRENT CONTROLLER DESIGN

The aim is to design of the control inputs i_{dq}^* by proposed method that guarantees the convergence of the sliding surface. These results in the desired dynamics of the PMSG model are expressed as follows:

$$v_{dq} = R_{dq} i_{dq}^* + \dot{\psi}_{dq} + \psi_{dq} p \omega_m^* \quad (18)$$

$$J \dot{\omega}_m^* = T_m - T_e^* - f_{jv} \omega_m^* \quad (19)$$

$$T_e^* = \frac{2}{3} p \psi_{dq}^T i_{dq}^* \quad (20)$$

where, T_e^* denotes the desired electromagnetic torque and ω_m^* represents speed of the turbine (desired speed). Let us define the flux error given bellow:

$$e_f = \begin{bmatrix} e_{fd} \\ e_{fq} \end{bmatrix} = \psi_{dq}^* + \psi_{dq} \quad (21)$$

where, $\psi_{dq}^* = [\psi_d^* \ \psi_q^*]^T$ denotes the desired value of the flux linkages vector. By Substituting (21) in (19) and considering the Lyapunov function $V_L(e_f) = 0.5 e_f^T e_f$, the control signals i_{dq}^* which guarantees the asymptotic convergence of the error e_f , is expressed as expressed as follows:

$$i_{dq}^* = -\frac{1}{R_{dq}} \left(K e_{\psi} - \left(\dot{\psi}_{dq}^* + p \omega_m \Im \psi_{dq}^* \right) \right) \quad (22)$$

where, $K = \begin{bmatrix} K_a & 0 \\ 0 & K_b \end{bmatrix}$, $K_a > 0$ and $K_b > 0$. The proof of the flux tracking error's exponential stability is given as follows: The time derivative along (19) of $V_L(e_f)$, yields the following relation:

$$\dot{V}_L(e_f) = -e_f^T K e_f \leq -\lambda_{\min} \{K\} \|e_f(t)\|^2, \forall t \geq 0 \quad (23)$$

$\lambda_{\min} \{K\}$ is the matrix and K represents the eigenvalues. The standard Euclidian norm's square of e_f is expressed as below:

$$\|e_f\|^2 = e_{fd}^2 + e_{fq}^2 \quad (24)$$

Combining (24) with $V_L(e_f)$, yields:

$$V_L(e_f) = 0.5 e_f^T e_f \leq \|e_f(t)\|^2, \forall t \geq 0 \quad (25)$$

Multiplying both sides of (25) with $(-\lambda_{\min} \{K\})$, yields:

$$(-\lambda_{\min} \{K\}) V_L(e_f) \geq (-\lambda_{\min} \{K\}) \|e_f(t)\|^2, \forall t \geq 0 \quad (26)$$

By combining (23) with (26), one obtains the following relation:

$$\dot{V}_L(e_f) \leq (-\lambda_{\min} \{K\}) V_L(e_f), \forall t \geq 0 \quad (27)$$

Integrating both sides of (27), gives:

$$V_L(e_f) \leq V_L(0) e^{-\lambda_{\min} \{K\} t}, \forall t \geq 0 \quad (28)$$

From (25) at $t = 0$, and multiplying it by $e^{-\lambda_{\min} \{K\} t}$, we get:

$$V_L(0) e^{-\lambda_{\min} \{K\} t} \leq \|e_f(0)\|^2 e^{-\lambda_{\min} \{K\} t} \quad (29)$$

Combining (29) with (28), yields:

$$V_L(e_f) \leq \|e_f(0)\|^2 e^{-\lambda_{\min} \{K\} t}, \forall t \geq 0 \quad (30)$$

From (25) and (30) it gives:

$$\|e_f(t)\|^2 \leq \|e_f(0)\|^2 e^{-0.5 \lambda_{\min} \{K\} t} \quad (31)$$

Therefore, with a rate of convergence $\lambda_{\min} \{K\}$ the error e_f is exponentially decreasing, thus the system is asymptotically stable.

Remark 2: It is preferable to choose a high but limited value for this gain K , to permits a good convergence rate of the parameter $\lambda_{\min} \{K\}$ and to avoid divergence of i_{dq} . This limitation can be realized by simulation tests.

D. DESIRED TORQUE COMPUTATION USING THE PROPOSED ADAPTIVE FO-PID

The PMSG works at maximum torque if the direct current is maintained to zero. Thus, the flux ψ_d is reduced to the flux ϕ_f created by the permanent magnet. Then, the desired flux linkages are given as bellow:

$$\psi_{dq}^* = \begin{bmatrix} \psi_d^* \\ \psi_q^* \end{bmatrix} = \begin{bmatrix} \phi_f \\ L_d i_d^* \end{bmatrix} \quad (32)$$

From (32) and (20), the desired flux along q-axis is given by the following expression:

$$\psi_q^* = \frac{2}{3} \frac{L_q}{p\phi_f} T_e^* \quad (33)$$

Note that the controller law has two parts: The term that encloses the reference dynamics and the damping term that makes the system strictly passive.

The reference torque is derived from the mechanical dynamic equation (19), which is expressed by the following relation:

$$T_e^* = J\dot{\omega}_m^* + T_m - f_{fv}\varepsilon_m \quad (34)$$

Here $\varepsilon_m = (\omega_m^* - \omega_m)$ represents the speed error. The objective is to minimize the speed error between the PMSG and the marine current turbine. The above Eq. (39) shows clearly that T_e^* is dependent on two factors: It is open loop and the convergence property is a function of (J, f_{fv}) [4], [17]. To overcome the two factors, a fractional order fuzzy PID controller is adopted. It is well known that the robustness of the FO-PID is much better than the traditional PID and PI loops [17]. The inputs of the adaptive FO-PID fuzzy controller are ε_m and its derivative $\Delta\varepsilon_m$, and the outputs are the FO-PID control gains k_p, k_i and k_d . Then, Eq. (34) is expressed by the following form:

$$T_e^* = J\dot{\omega}_m^* - k_p\varepsilon_m - k_i D_t^{-\alpha} \varepsilon_m - k_d D_t^{\beta} \varepsilon_m \quad (35)$$

Here $K_p > 0, K_d > 0$ and $K_i > 0$, while $D_t^{-\alpha}$ represents fractional integration of order α and D_t^{β} represents fractional derivative with order β . For more information about the definition and theory of the FO-PID controller, the reader is referred to the following references [27], [29]. The design process of the fuzzy controller is the same as that used in section 3.A. The design of T_e^* by the adaptive FO-PID is illustrated by Fig. 5.

IV. GLOBAL STABILITY OF THE PROPOSED STRATEGY

Lemma 2: The system is passive only if the dynamics expressed in (22) given by $\frac{1}{R_{dq}} (\dot{\psi}_{dq}^* + p\omega_m \Im \psi_{dq}^*)$ and ψ_{dq} are considered as input and output.

Proof: From (21), (22) and (6), the following equation is deduced:

$$\dot{\psi}_{dq} + \psi_{dq} p\omega_m = -R_{dq} \frac{1}{R_{dq}} (\dot{\psi}_{dq}^* + p\omega_m \Im \psi_{dq}^*) - Ke_f \quad (36)$$

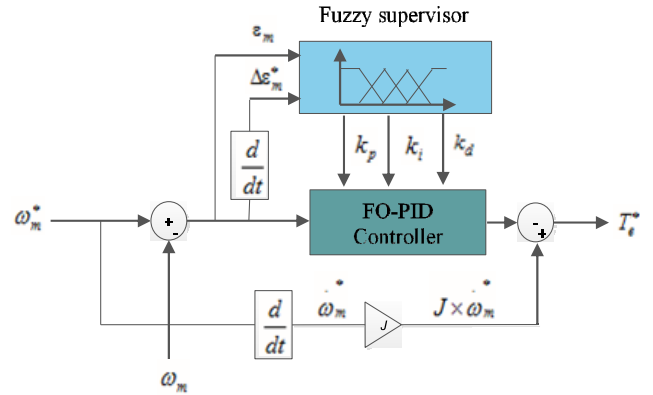


FIGURE 5. Desired torque with adaptive FO-PID.

Multiplying (36) by $\frac{\psi_{dq}^T}{R_{dq}}$, yields the following expression:

$$\psi_{dq}^T \vartheta = -\frac{1}{2R_{dq}} \frac{d(\psi_{dq}^T \psi_{dq})}{dt} - \psi_{dq}^T Ke_f \quad (37)$$

where, $\vartheta = \frac{1}{R_{dq}} (\dot{\psi}_{dq}^* + p\omega_m \Im \psi_{dq}^*)$. Since $\psi_{dq}^T \psi_{dq} = 0$, the term $p\omega_m R_{dq}^{-1} \psi_{dq}^T \psi_{dq}$ has no relation with the right hand side of Eq (37). From (31), e_f decreases exponentially. Then, the term $\psi_{dq}^T Ke_f$ is insignificant, and (37) become as bellow:

$$\psi_{dq}^T \vartheta = -\frac{1}{2R_{dq}} \frac{d(\psi_{dq}^T \psi_{dq})}{dt} \quad (38)$$

Integration of (38), gives the following epression:

$$\int_0^t \psi_{dq}^T \vartheta dt = -\frac{1}{2R_{dq}} (\psi_{dq}^T \psi_{dq})(t) + \frac{1}{2R_{dq}} (\psi_{dq}^T \psi_{dq})(0) \quad (39)$$

Thus, by introducing $V_L(e_f)$, Eq. (39) can be rewritten as given below:

$$\int_0^t \psi_{dq}^T \vartheta dt = -\frac{1}{R_{dq}} V_L(t) + \frac{1}{R_{dq}} V_L(0) \quad (40)$$

Eq. 40, proves the passivity of the system while the energy balance of the system is independent of the term $p\omega_m R_{dq}^{-1} \psi_{dq}^T \psi_{dq}$, which has no influence on the stability of the system. Thus, the closed-loop system is globally stable.

V. PI CONTROLLER STRUCTURE OF THE GSC

To control reactive power support and stabilize the DC link voltage, the GSC converter controller is utilized. From [17], [20] the dynamics of GSC converter are expressed as follows:

$$\begin{bmatrix} \dot{V}_{id} \\ \dot{V}_{iq} \end{bmatrix} = R_f \begin{bmatrix} i_{df} \\ i_{qf} \end{bmatrix} + \begin{bmatrix} L_f \dot{i}_{df} - \omega L_f i_{qf} \\ L_f \dot{i}_{qf} - \omega L_f i_{df} \end{bmatrix} + \begin{bmatrix} V_{gd} \\ V_{gq} \end{bmatrix} \quad (41)$$

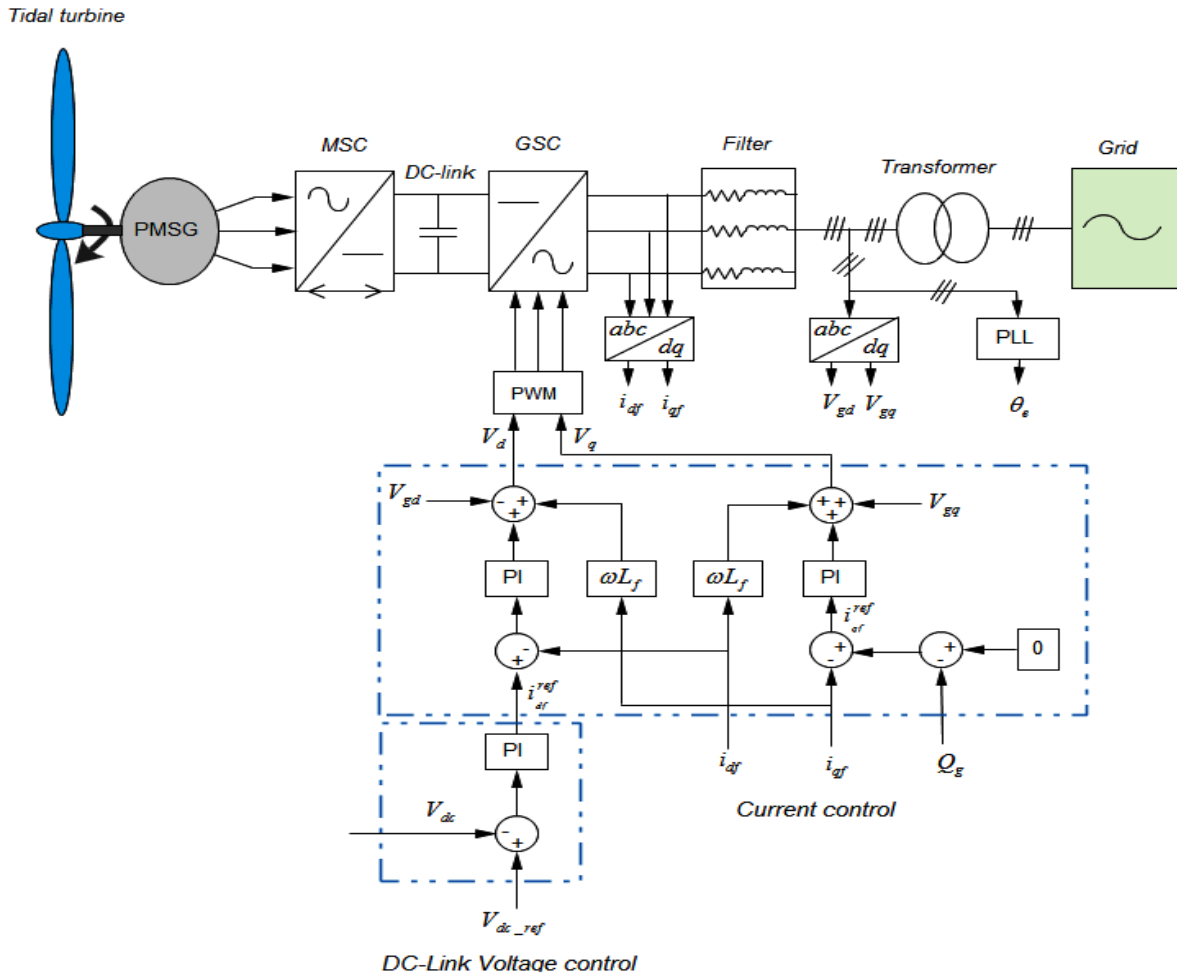


FIGURE 6. GSC PI controller structure.

From Eq. 41, ω is the grid frequency, V_{gd}, V_{gq} show the dq voltages, V_{id}, V_{iq} are the voltages measured at the inverter terminal, L_f represents filter inductance, i_{df} and i_{qf} are the dq component of the grid currents, and R_f gives resistance of the filter.

Dynamics of DC link are expressed as follows:

$$C\dot{V}_{dc} = \frac{3}{2} \frac{V_{gd}}{V_{dc}} i_{df} - i_{dc} \quad (42)$$

From Eq. 42, C is capacitance of DC link, V_{dc} represents voltage across DC link, and i_{dc} represents DC current.

Expressions for reactive and active powers are given by the following equations:

$$\begin{cases} P_g = \frac{3}{2} v_{gd} i_{df} \\ Q_g = \frac{3}{2} v_{gd} i_{qf} \end{cases} \quad (43)$$

The PI controller of the DC-link is expressed as follows:

$$i_{df}^{ref} = k_{dcp} (V_{dc_ref} - V_{dc}) - k_{dci} \int_0^t (V_{dc_ref} - V_{dc}) d\tau \quad (44)$$

where, $k_{dcp} > 0$ and $k_{dci} > 0$. The PI loops of the currents are given as follows:

$$v_{gd}^{PI} = k_{gp}^d (i_{df}^{ref} - i_{df}) - k_{gi}^d \int_0^t (i_{df}^{ref} - i_{df}) d\tau \quad (45)$$

$$v_{gq}^{PI} = k_{gp}^q (i_{qf}^{ref} - i_{qf}) - k_{gi}^q \int_0^t (i_{qf}^{ref} - i_{qf}) d\tau \quad (46)$$

where $k_{gp}^d > 0, k_{gi}^d > 0, k_{gp}^q > 0$ and $k_{gi}^q > 0$.

VI. NUMERICAL RESULTS

In this part, extensive numerical investigations are performed using Matlab/Simulink. System parameters are listed in Table 2. The reference reactive power is fixed to zero and the DC-link reference is set to 1150V. Using the pole location method, the damping parameters are given as follows: The PI gains of DC-link loop are chosen as: $k_{dcp} = 5$ and $k_{dci} = 500$. The damping gain is $K = 400$. Gains of the current loop controller are $k_{gp}^d = k_{gp}^q = 9$ and $k_{gi}^d = k_{gi}^q = 200$. The values of the two coefficients of FO-PID regulator are chosen as $\alpha = -0.79$ and $\beta = +0.55$. For more information about

TABLE 2. System parameters.

PMSG parameter	Value
Tidal density (ρ)	1024 kg/m ³
Tidal turbine radius (R)	10 m
Stator inductance (L_{dq})	0.3 mH
Stator resistance (R_s)	0.006 Ω
Stator inductance (L_{dq})	0.3 mH
Pole pairs number (p)	48
Flux linkage (ϕ_f)	1.48 Wb
Total inertia (J)	35000 kg.m ²
DC-link voltage (V_{dc})	1150 V
DC-link capacitor (C)	2.9 F
Grid-filter resistance (R_f)	0.3 pu
Grid-filter inductance (L_f)	0.3 pu

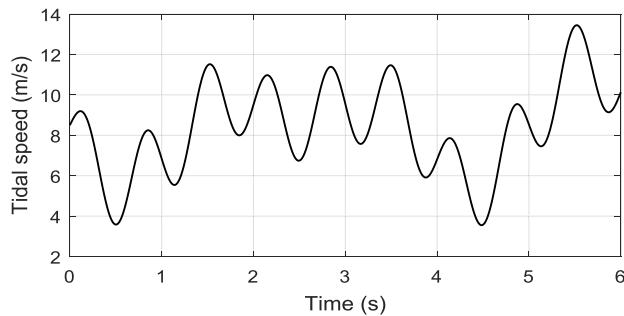


FIGURE 7. Tidal speed.

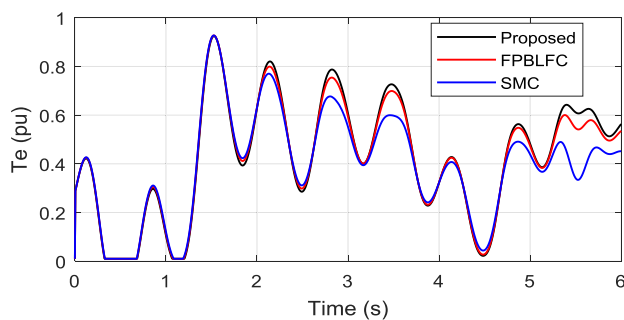


FIGURE 8. Electromagnetic torque.

the selection process of these two parameters, the reader is referred to [29]. The proposed strategy will be compared to the FPBLFC controller [17] and the second-order sliding mode control (SMC) controller [7]. Two scenarios are discussed: The first one deal with the testing of the proposed control under swell effect and with ideal parameters of the system. In the second scenario, the system is tested under swell effect and parameter uncertainties.

A. FIXED PARAMETERS ANALYSIS

The velocity of the tidal current with the considered swell effect is presented in Fig. 7. The tidal speed used in the simulation study varies between 3.9 m/s to 13.8 m/s. Fig. 8 shows the comparison of the electromagnetic torque of the PMSG generator produced as a result of the tidal velocity

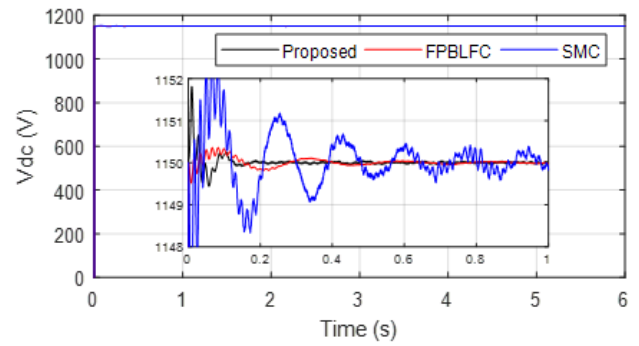


FIGURE 9. DC-link voltage.

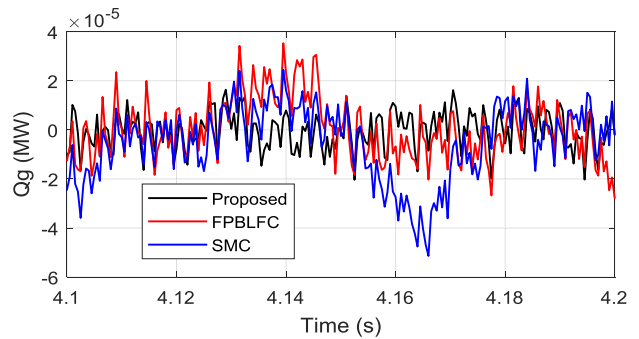


FIGURE 10. Zoom on reactive power.

of Fig. 7. One clearly concludes that under the influence by the swell effect the proposed strategy shows a higher electromagnetic torque as compared to the FPBLFC and SMC methods. From Fig.8 and in the time interval between $t = 2-6$ seconds, it is very obvious that the average generated electromagnetic torque with the proposed control scheme is higher as compared to the FPBLFC and SMC control methods. Fig. 9 shows the DC link voltage tracking response with FPBLFC, SMC and proposed control methods. From the presented results, a transient overshoot of +2 volts and undershoot of -2 volts is observed with SMC controllers. Similarly, a voltage overshoot of +0.5 volts and undershoot of -0.5 volts are observed with the FPBLFC control scheme. In case of proposed control method, lowest overshoot and undershoots are observed in the tracking response of the DC- link voltage. With the proposed control scheme, voltage overshoot of +0.3 volts and undershoot of -0.3 volts are recorded. Moreover, the proposed control ensures small oscillation and fast voltage tracking response. Fig. 10 shows the reactive power tracking comparison with SMC, FPBLFC and the proposed control schemes. From the presented results, a peak error of 0.5×10^{-4} is observed with SMC, 0.4×10^{-4} with the FPBLFC and 0.2×10^{-4} with the proposed controller. Although the reactive power tracking errors are well bounded with all variants of control schemes, however, the proposed controller ensures the lowest tracking error and fast error tracking response. The active power response is shown in Fig. 11. From the presented results, it is obvious that in the time

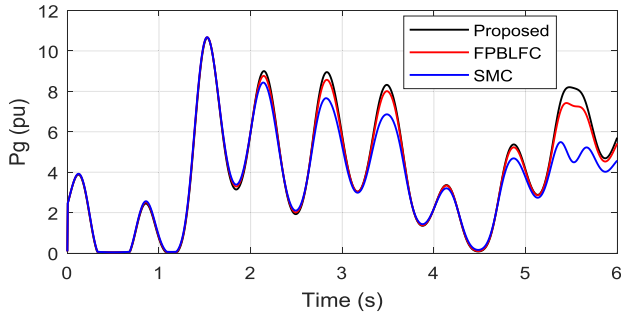


FIGURE 11. Active power.

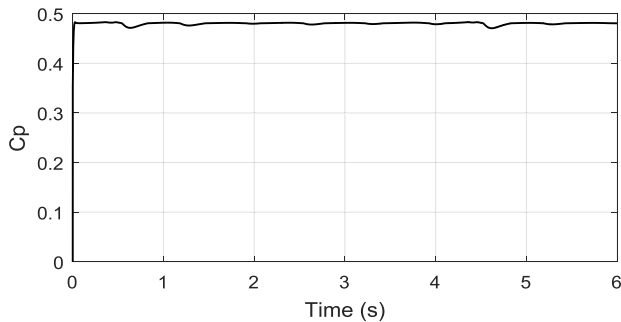


FIGURE 12. Power coefficient.

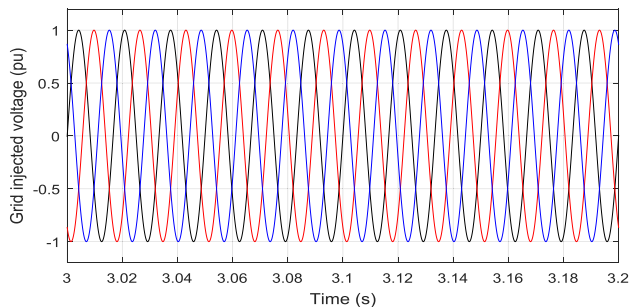


FIGURE 13. Generated voltage.

duration $t = 2-6$ seconds, the proposed controller integrates higher average active power to the grid under the swell effect. Fig. 12 shows the power coefficient of the turbine. Fig. 13 illustrates the grid injected voltage and from the presented results, it is obvious that the proposed control method ensures perfect sine voltage injection to the grid. In summary, and in comparison, to the other two benchmark nonlinear control methods, the proposed control scheme ensured lowest tracking errors, higher electromagnetic torque and active power with fast convergence of the states under swell effect.

B. ROBUSTNESS ANALYSIS

In the previous section, we presented fixed parameter analysis for the discussed control schemes under swell effect. In this section, the robustness of the proposed control scheme is tested under the effects of parameter variations and swell effects. Fig. 14 shows the response of the electromagnetic torque with the proposed control scheme under the effect

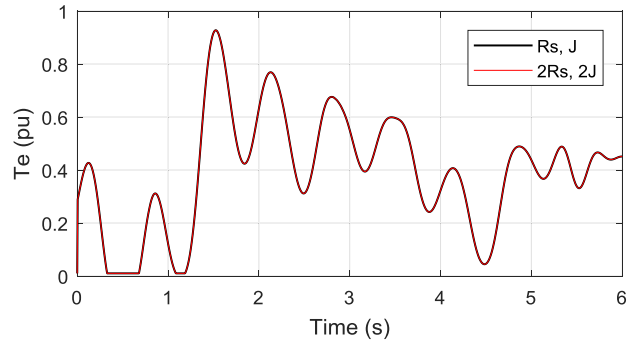


FIGURE 14. Torque response due to disturbances.

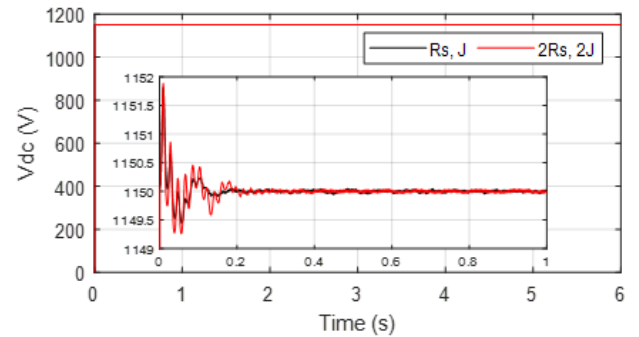


FIGURE 15. DC-link response due to disturbances.

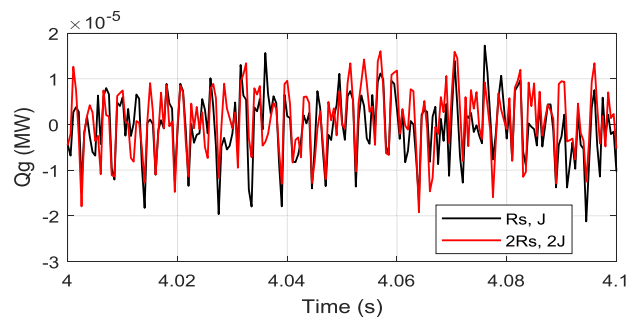


FIGURE 16. Reactive power response due to disturbances.

of parameters variations. Simultaneous variations of +100% in the stator resistance and +100% in the total inertia are considered. From the presented results, it is obvious that the proposed controller efficiently compensates the parametric changes and the swell effect. Fig. 15 shows DC-link voltage regulation response with the proposed control scheme and under the above-mentioned disturbances. The presented results confirm the robustness of the proposed control scheme to the above-mentioned disturbances and the voltage regulation response is comparable to the fixed parameter test case. The measured error in DC-link voltage is approximately ± 0.3 volts, which is comparable to test 1 (see Fig. 9). Fig. 16 confirms the robustness of the proposed strategy for tracking response of the reactive power to its reference value under the variations. From the presented results, it is observed that the recorded error is $\pm 0.2e-4$, which is approximately the

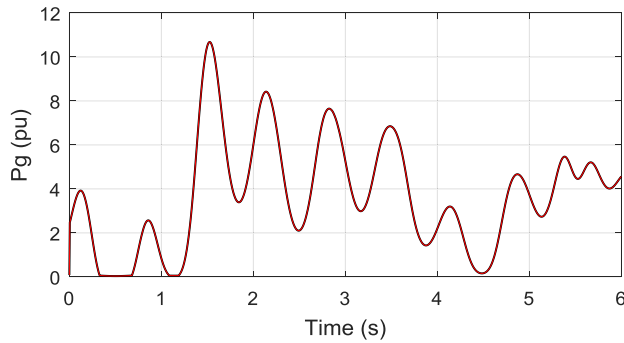


FIGURE 17. Active power response due to disturbances.

same as test case 1. Fig. 17 shows the active power transmitted to the grid under the effect of parameters variation and swell effect. It is obvious from the presented results that under these changes the active power is not influenced. In summary, under the effect of parametric uncertainty, the proposed control ensured stable operation of the DC-link voltage and the power transmitted to the grid. This further shows that the proposed controller can gain the stability performances with a constant power loads under parameter uncertainties and external disturbances studied in [30] and [31], for an efficient and secure electricity production [33], [34]. One can conclude after this test, that the proposed control firmly resists against simultaneous parameter changes and swell effect. Thus, in each scenario, the proposed strategy presents a higher dynamic response and robustness as compared to the other two nonlinear control methods. It also validates the mathematical demonstrations of the global stability and the exponential convergence of the error.

VII. CONCLUSION

In this paper, a novel intelligent passivity-based modified super twisting algorithm is presented for a PMSG assisted marine current conversion system. The proposed strategy is adopted to extract the maximum power from the tidal current, taking into account the entire dynamics of the PMSG when synthesizing the controller. The adaptive FO-PID is selected to guarantee fast response of the PMSG, and operate it at the optimal dynamics. Numerical simulations are presented under disturbances like swell effect and parameter changes. A comparative analysis of the system performance with proposed strategy is presented. All drawbacks of the conversion system are addressed and the control objectives are well achieved. The developed control strategy shows superior performance and higher robustness in comparison to the other nonlinear strategies. Future works will be focused on the experimental validation of the proposed controller.

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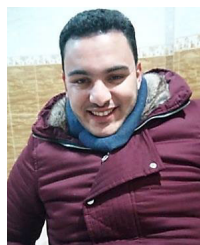


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