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Nonconvex Wavelet Thresholding Total Variation Denoising Method for Planetary Gearbox Fault Diagnosis

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ABSTRACT The vibration-based analysis is an effective technique for planetary gearbox fault diagnosis, but a difficult task is how to accurately identify fault features from noisy vibration signals. In this paper, a nonconvex wavelet thresholding total variation (WATV) denoising method is proposed for planetary gearbox fault diagnosis, which combines wavelet-domain sparsity and total variation (TV) regularization. The TV regularization algorithm is employed to modify the retained wavelet coefficients so that the occurrence of oscillations caused by wavelet thresholding is suppressed. To overcome the underestimation shortcoming of L1-norm regularization, nonconvex penalty function regularization is used to strongly promote the sparsity of estimation while guaranteeing that the global optimal solutions are obtained even though the objective function is nonconvex. Then, the split augmented Lagrangian shrinkage (SALSA) method is developed to solve the nonconvex WATV denoising problem. Two experimental studies are performed to verify the performance and effectiveness of the proposed method. Comparisons with the soft thresholding and basis pursuit denoising (BPD) methods show that the proposed method can accurately estimate the fault features from vibration signals, which means that the proposed method is an effective and promising tool for planetary gearbox fault diagnosis.

INDEX TERMS Planetary gearbox, fault diagnosis, nonconvex regularization, total variation, wavelet thresholding.

I. INTRODUCTION

Due to their small size, lightweight, large transmission ratio and strong load capacity, planetary transmission gearboxes are widely used in military aircraft, new armored vehicles, self-propelled artillery and other military equipment and civilian equipment. Due to the harsh operating conditions and intensive impact load of planetary gearboxes, gear faults such as cracks and missing teeth frequently occur [1], [2]. Failure of planetary gearbox seriously affects the safety and reliability of the equipment. If the fault is not diagnosed in time, it will cause secondary damage to the equipment and result in major economic losses and even human casualties. Thus, fault diagnosis [3] and fault trend forecasting [56] have attracted considerable attention in the past decades.

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Vibration-based analysis is an effective method for diagnosing mechanical faults [3]–[6], because the vibration signals generated by mechanical faults indicate the machinery operation condition. The information contained in the vibration signal describes not only the machine health condition but also the severity. However, in a planetary gearbox, the transmission system is more complicated than a fixed-axis gear, so the transmission paths from gear meshing points to transducers are multiple and time-varying, which may deteriorate or attenuate the vibration response of faulty components [3], [7]. In addition, the vibration signals are usually degraded by strong background noise from the working environment and other machine components. Thus, how to accurately extract the fault features from a complex noisy signal is a difficult task for planetary gearbox fault diagnosis.

Various signal processing techniques have been widely applied to vibration signal analysis and fault diagnosis during the past decades, such as time-frequency analysis [8]–[10],

the spectral kurtosis (SK) [11]–[13], the empirical mode decomposition (EMD) [14]–[16], the local mean decomposition (LMD) [17] and the wavelet transform (WT) [18]–[20]. The multiresolution analysis ability of the WT makes it suitable for extracting fault features from nonstationary vibration signals. Wang *et al.* [21] proposed a method combining the Morlet wavelet and correlation filtering to identify the impulse response parameters and the cyclic period between adjacent impulses, which is an effective way to extract features of gearbox fault diagnosis. In [22], the wavelet-based multiscale slope features were extracted from the slope of logarithmic variances calculated from the wavelet coefficients of the discrete WT, and then the extraction features were used to classify gearbox faults with high accuracy and stability. He *et al.* [23] presented a wavelet packet base-selection method that selected an optimal set of time-frequency subspaces to produce discriminant features to enhance the accuracy of gearbox fault diagnosis. Wang *et al.* [24] constructed a new multiwavelet via an adaptive lifting scheme to extract fault features from vibration signals of a gearbox. To overcome the disadvantage that the common threshold denoising method misses important but weak features in gear fault diagnosis, Yuan *et al.* [25] developed a novel method combining customized multiwavelet lifting schemes with sliding window denoising. In [26], a series of sets of wavelet packet coefficients on various frequency bands were taken as an input of the deep residual networks to improve the performance of planetary gearbox fault diagnosis.

Unfortunately, most of the previous studies have chosen standard and fixed wavelet basis functions independent of the given signal. In practice, the vibration signals are the dynamic integrated response of running systems, fault components, transmission paths and so on. Even the same faults generate various dynamic response signals in different transmission paths. Thus, fixed wavelet basis functions independent of the input dynamic response signals tend to reduce the accuracy of fault diagnosis [27]. On the other hand, universal thresholds are usually set in previous denoising methods, which are the same for the decomposition coefficients in the same layer. The universal thresholds often obscure the decomposition coefficients and may lead to the loss of some critical but relatively weak information in the fault feature detection, possibly reducing the accuracy of fault diagnosis [28].

Recently, a new branch of signal processing method, sparse representation has received considerable attentions in the field of mechanical fault diagnosis. The application of sparse representation in mechanical fault diagnosis was initially studied in [29]. To extract fault features from gearbox vibration signals, Li *et al.* [30] proposed a novel multiple enhanced sparse decomposition (MESD) method to address multiple feature extraction for gearbox compound fault vibration signals. In [31], based on the oscillatory behavior of the vibration signal, a sparsity-enabled signal decomposition method using L1-norm regularization was proposed for fault feature extraction of gearboxes. For multi-fault diagnosis of gearboxes, Zhang *et al.* [32] introduced a resonance-based sparse signal

decomposition method with a comb filter. Yu *et al.* [33] proposed an improved morphological component analysis method which is a sparsity-based decomposition method, to diagnosis compound faults in a gearbox. The above sparse representation methods have satisfactory results for feature identification in gearbox fault diagnosis. In these studies, the classical L1-norm regularization is used to regularize the sparse representation problem, because the L1-norm induces the sparsity of the estimation more effectively than other convex penalties [34]. However, L1-norm solutions are not ideal for planetary gearbox fault diagnosis, because L1-norm regularization often underestimates the high-amplitude components of interest, which may still make the weak fault components obscured in noise and hard to be effectively identified [34]–[36].

To overcome the underestimation characteristic and enhance the sparsity of the estimation, many nonconvex sparse regularization methods that replace the classical L1-norm regularization with nonconvex sparsity-inducing penalties have been developed and used in mechanical fault diagnosis. To improve the decomposition accuracy for gearbox fault diagnosis, Cai *et al.* [35] introduced an improved sparsity-enabled signal decomposition method which used the generalized minimax-concave penalty function as a nonconvex regularizer to enhance the sparsity in the sparse approximation. In the STFT domain, Ding *et al.* [37] used a nonconvex penalty function to promote the sparsity of group STFT domain coefficients in an optimization problem, allowing the periodically oscillatory fault features of rotating machinery to be effectively extracted. In [36], to address the fault feature enhancement for wind turbine gearbox vibration signals, Wang *et al.* proposed a dual-enhanced sparse decomposition method in which the nonconvex generalized minimax-concave penalty was used to construct the sparse regularized cost function. He *et al.* [38] extended the overlapping group sparsity to the nonconvex regularization problem, the proposed method used a nonconvex penalty function to model the periodicity of the sparse groups, making this method suitable for feature extraction in machinery fault diagnosis.

The sparse enhancement properties of the nonconvex penalty function provide a new insight on how to accurately extract fault features. In this paper, we propose a nonconvex wavelet thresholding total variation denoising method for planetary gearbox fault diagnosis, which combines wavelet-domain sparsity and total variation (WATV) regularization. The proposed method employs the TV regularization algorithm to modify the retained wavelet coefficients so that the restoration process does not degrade the information that has been considered as significant in the denoising step. Moreover, to strongly promote the sparsity of estimation, a nonconvex penalty function is employed as the regularizer of the sparse representation, and this function is guaranteed to obtain the global optimal solution even though the objective function is nonconvex. We present two experimental studies to verify the effectiveness of the proposed

method in the diagnosis of localized faults in a planetary gearbox. Comparisons show that the proposed method can significantly improve the accuracy of planetary gearbox fault diagnosis compared with the results of the soft thresholding denoising and basis pursuit denoising (BPD) method.

The paper is organized as follows. Section 2 illustrates the basic preliminaries. Section 3 presents the non-convex WATV denoising model, the nonconvex penalty functions and convexity condition, and the split augmented Lagrangian shrinkage (SALSA) method is derived to solve the non-convex WATV denoising problem. In Section 4, two experiment studies are performed to verify the effectiveness of the proposed method. Finally, conclusions are summarized in Section 5.

II. PRELIMINARIES

Vibration signals observed from a faulty planetary gearbox can be modeled as

$$y(t) = x(t) + w(t) \tag{1}$$

where $y(t) \in R^N$ is the measured signal, which usually contains noise and fault signals; $x(t) \in R^N$ is the fault signal, which contains impulsive components with periodic characteristics caused by localized gearbox faults; and $w(t) \in R^N$ is white Gaussian noise. The challenging problem in the fault diagnosis of a planetary gearbox is how to accurately restore the fault signal $x(t)$ from the noisy measured signal $y(t)$.

A. WAVELET THRESHOLDING DENOISING

Generally, wavelet denoising contains three steps [38], [39]: forward transformation of the noisy signal to an orthogonal domain, reduction of the wavelet coefficients smaller than a given threshold, and inverse transformation of the data to the original domain. Let $\phi_{j,k}(t)$ be an orthogonal wavelet basis function, so that the standard WT of signal $y(t)$ can be written as

$$y(\vec{a}, t) = \sum_{j,k} a_{j,k} \phi_{j,k}(t) \tag{2}$$

where $\vec{a} = \{a_{j,k}\}$ are the corresponding wavelet coefficients defined as

$$a_{j,k} = \int y(t) \phi_{j,k}(t) dt \tag{3}$$

Because of its simplicity, the wavelet thresholding technique has been widely used in engineering practice since the beginning of wavelet use in signal processing. One way of describing wavelet thresholding technology is to establish a set $M = \{(j, k) \in K : |a_{j,k}| \geq \eta\}$ to record the indexes of the wavelet coefficients, then preserve all coefficients whose indexes belong to M and truncate the others to zero

$$\tau(a_{j,k}) = \begin{cases} a_{j,k} & \text{if } (j, k) \in M \\ 0 & \text{otherwise} \end{cases} \tag{4}$$

where $\tau(\cdot)$ is a thresholding operator. For example, if τ is a hard thresholding operator, M is defined as the set of all

coefficients whose values are larger than a given threshold. If τ is a linear thresholding operator, then M is taken as the index set of all low frequencies.

B. TOTAL VARIATION DENOISING

In signal denoising problems, estimation of a noise free signal $x(t)$ from an observed noisy signal $y(t)$ is an inverse problem as well as a classical ill-posed problem [40]. A standard and efficient way to deal with inverse problems is to define a suitable objective function $F(x)$ consisting of a data term (consistency with the measurements) and a regularization term (based on prior information), and then to find the signal minimizing $F(x)$ [41]. Good denoising results should provide a balance between the regularization term and the data term and can be achieved only with some form of regularization penalizing undesirable solutions. Accordingly, typical criteria have the form

$$\hat{x} = \arg \min_x \left\{ F(x) = \|y - x\|_2^2 + \lambda \phi(x) \right\} \tag{5}$$

where $\|y - x\|_2^2$ is a data fidelity term used to measure the difference between y and x , $\phi(x)$ is a penalty function (or regularization term) that should be chosen as to penalize unwanted behavior in x , and $\lambda > 0$ is the regularization parameter, used to balance the trade-off between two terms.

If the signal of interest is known to be sparse, this prior information can be used to come up with a sparse regularization method. In this case, the penalty function is defined to measure the number of nonzero values in x , i.e., $\phi(x) = \|x\|_0$ where $\|x\|_0$ is the l_0 -norm and defined as $\|x\|_0 = \sum_{n=1}^N |x_n|_0$. Unfortunately, with $\phi(x)$ defined as such, the regularization problem in (5) is an NP-hard problem for which the objective function $F(x)$ cannot readily to minimize [42], [43]. Therefore, it is common to replace the l_0 -norm by the l_1 -norm as the penalty function in practical applications, because it induces sparsity most effectively and does not sacrifice the convexity of the objective function. The l_1 -norm regularization problem is given by

$$\hat{x} = \arg \min_x \left\{ F(x) = \|y - x\|_2^2 + \lambda \|x\|_1 \right\} \tag{6}$$

where $\|x\|_1$ is the l_1 -norm defined as $\|x\|_1 = \sum_{n=1}^N |x_n|_1$. The l_1 -norm regularization defined in (6) is also known as BPD [41], and has been widely used in image denoising, fault diagnosis, and ultrasonic signal processing.

Another widely used nonlinear filter method based on sparse model is TV [44], [45], which operates under the assumption that the derivative of the underlying signal is sparse. It is defined as a convex optimization problem of minimizing the cost function comprising a nondifferentiable convex penalty term and a quadratic data fidelity term. The TV in the signal $x \in R^N$ is defined as

$$TV(x) = \sum_{n=1}^N |x_n - x_{n-1}|_1 = \|Dx\|_1 \tag{7}$$

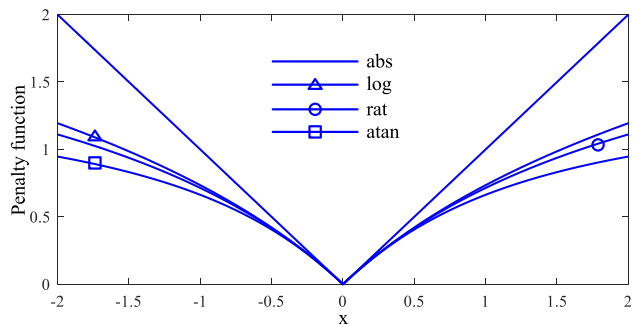


FIGURE 1. Example of penalty functions.

strictly convex. The objective function in (10) can be written as

$$F(a) = F_1(a) + F_2(a) \tag{12}$$

where

$$F_1(a) = \sum_{j,k} \left\{ \frac{1}{2} [(Wy)_{j,k} - a_{j,k}]^2 + \lambda_j \phi(a_{j,k}; b_j) \right\} \tag{13}$$

$$F_2(a) = \beta \|DW^T a\|_1 \tag{14}$$

Note that if F_1 and F_2 are strictly convex, then F is strictly convex. F_2 is defined as the l_1 -norm, so it is a convex function. Hence, it suffices to find the parameter b such that F_1 is strictly convex.

Combining the propositions 1 and 2 in [53], we note the following proposition.

Proposition 1: Let $\phi(x; b)$ be a nonconvex penalty that satisfies the above-listed properties, with $b \geq 0$. The function $G(x)$ defined as

$$G(x) = \frac{1}{2} (y - x)^2 + \lambda \phi(x; b) \tag{15}$$

is strictly convex if

$$0 < b < \frac{1}{\lambda} \tag{16}$$

Proof: By the Lemma A in [53], to ensure that $G(x)$ is strictly convex, the second derivative of $G(x)$ must be positive on $R/\{0\}$ and $G'(0^-) < G'(0^+)$. The first derivative of $G(x)$ is

$$G'(x) = x - y + \lambda \phi'(x; b) \tag{17}$$

By the property 5 in the part B of the part III, $G(0^+) = -y + \lambda$; by symmetry $G'(0^-) = -y - \lambda$ and $\lambda > 0$, $G'(0^-) < G'(0^+)$. To ensure that $G''(x)$ is positive on $R/\{0\}$, we have the following condition:

$$G''(x) = 1 + \lambda \phi''(x; b) > 0 \text{ for } x > 0 \tag{18}$$

or

$$\phi''(x; b) > -\frac{1}{\lambda} \tag{19}$$

By the property 6 in the part B of the part III, we have $b > 1/\lambda$. Hence, F_1 is strictly convex if and only if

$$0 < b < \frac{1}{\lambda} \tag{20}$$

Then, F_1 being a sum of strictly convex G is strictly convex. It follows that the objective function F in (10), being the sum of F_1 (strictly convex) and F_2 (convex), is strictly convex, with the condition

$$0 < b_j < \frac{1}{\lambda_j} \tag{21}$$

D. SALSA ALGORITHM FOR NONCONVEX WATV

With the convexity condition (21), we can reliably obtain via convex optimization the global minimum of (10) as long as the parameter b is chosen to satisfy (21). In this paper, the split augmented Lagrangian shrinkage (SALSA) [55] method is used to solve the nonconvex WATV denoising problem. The good convergence of SALSA has been proven in practice [42], [55].

With variable splitting, problem (10) can be transformed into the constrained optimization problem

$$\begin{aligned} & \min_{a,c} f_1(a) + f_2(c) \\ & \text{subject to } a = c \end{aligned} \tag{22}$$

where

$$f_1(a) = \|Wy - a\|_2^2 + \sum_{j,k} \lambda_j \phi(a_{j,k}, b_j) \tag{23}$$

$$f_2(c) = \beta \|DW^T c\|_1 \tag{24}$$

Using the augmented Lagrangian method to represent the problem (24), we have

$$L(a, c, \mu) = f_1(a) + f_2(c) + \frac{\mu}{2} \|a - c - d\|_2^2 \tag{25}$$

where d is a multiplier vector to constraint $a = c$. The solution of (25) can be obtained by iteratively minimizing with respect to each variable alternately, as proven in [55]. Each iteration step of the SALSA algorithm is given by

$$a_{k+1} = \arg \min_a f_1(a) + \frac{\mu}{2} \|a - c - d\|_2^2 \tag{26}$$

$$c_{k+1} = \arg \min_c f_2(c) + \frac{\mu}{2} \|a - c - d\|_2^2 \tag{27}$$

Substituting $f_1(a)$ and $f_2(c)$ into (26) and (27) respectively, the explicit form of the SALSA algorithm can be obtained as follows:

$$\begin{aligned} a_{k+1} = \arg \min_c & \|Wy - a\|_2^2 + \sum_{j,k} \lambda_j \phi(a_{j,k}, b_j) \\ & + \frac{\mu}{2} \|a - c - d\|_2^2 \end{aligned} \tag{28}$$

$$c_{k+1} = \arg \min_c \beta \|DW^T c\|_1 + \frac{\mu}{2} \|a - c - d\|_2^2 \tag{29}$$

$$d_{k+1} = d_k - (a_{k+1} - c_{k+1}) \tag{30}$$

Table 1 summarizes the whole algorithm for solving problem (10), which is guaranteed to converge to the unique

TABLE 1. Salsa algorithm for the nonconvex WATV denoising problem.

SALSA Algorithm
1. Input: y, λ_j, β
2. Set: $k = 0, d = 0, a = 1/\lambda_j$
3. Repeat:
4. $a_{k+1} = \arg \min_c \ Wy - a\ _2^2 + \sum_{j,k} \lambda_j \phi(a_{j,k}, b_j) + \frac{\mu}{2} \ a - c - d\ _2^2$
5. $c_{k+1} = \arg \min_c \beta \ DW^T c\ _1 + \frac{\mu}{2} \ a - c - d\ _2^2$
6. $d_{k+1} = d_k - (a_{k+1} - c_{k+1})$
7. $k \leftarrow k + 1$
8. Until the stopping criterion is satisfied.
9. Return: $x = W^T a$

global minimizer. Both minimization problems, namely, iteratively thresholding and TV denoising, can be solved exactly. By running SALSA until the stopping criterion (the relative variation of the objection function falls below 10^{-5}) is satisfied, the global optimal solution of the nonconvex WATV denoising problem can be found, as shown in Fig. 2.

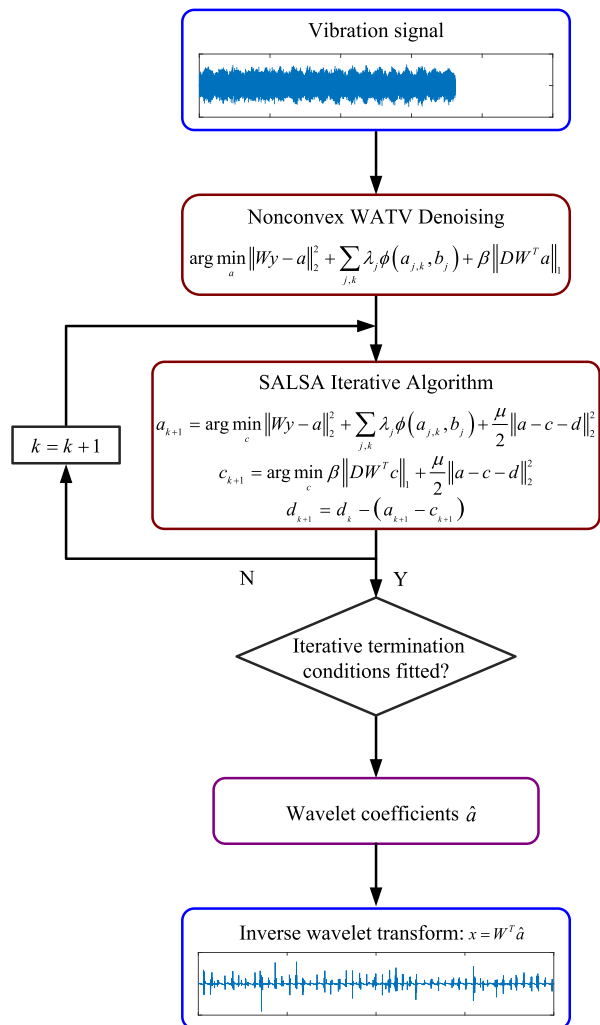


FIGURE 2. Flow chart of the nonconvex WATV denoising method.

IV. EXPERIMENTAL VERIFICATION

A. EXPERIMENTAL SETUP

To verify the effectiveness of the proposed nonconvex WATV denoising method in the fault diagnosis of a planetary gearbox, a gearbox test rig is established and experiments on it are carried out. The experimental object is a three-shaft, clutch-shifting and three-degree-of-freedom planetary gearbox. It has three fixed-shaft gear trains and three planetary gear trains, and the main shaft gear and the intermediate shaft gear are fixed-axis gears. There are three planetary rows on the main shaft: K1, K2 and K3. K1 is an external meshing double planetary row, and the other two are simple planetary rows. According to the structure characteristics and working principle of planetary gearbox, a fault simulation test rig for the planetary gearbox is designed as shown in Fig. 3. The test rig is composed mainly of a driving motor, transmission box, clutch, planetary gearbox, loading motor, hydraulic station, and speed and torque instrument and test system.

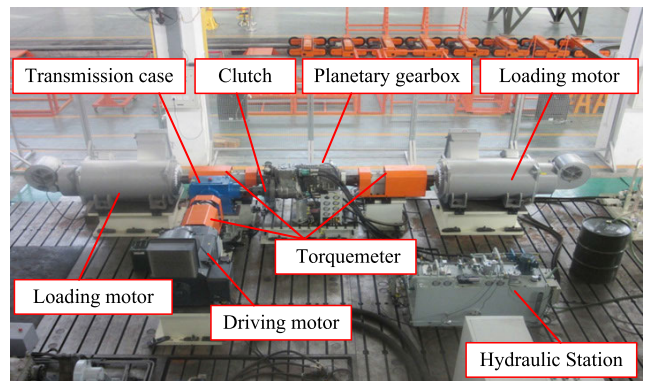


FIGURE 3. A planetary gearbox test rig.

The driving motor transmits power to the planetary gearbox through the gearbox and clutch, and then the power is transferred to the load generator through the gearbox. The driving motor provides power for the planetary gearbox, and the speed of the motor is controlled by a speed controller, which accommodates adjustment in the range of 0-1500 r/min. The load is provided by the loading motor connected to the output shaft of the planetary gearbox, and the load torque can be adjusted by controlling the platform in the range of 0-900 N•m. The hydraulic station is responsible for providing lubricant oil pressure and shift pressure for the planetary gearbox.

B. EXPERIMENTAL SCHEME

In a planetary gearbox, sun gear teeth are the most vulnerable part to fail, since their multiplicity of meshes with the planet gear increases the possibility of damage to the sun gear. In this paper, faults of broken teeth and crack on the sun gear in the K3 row are considered because broken teeth and cracks are common fault types in gearboxes. In practice, cracks usually begin at the maximum stress point of the gear teeth and then develops along the normal line of the root curve. Thus, the

cracks are processed along the normal line of the tooth's root curve by machine tools in our experiments. In addition, the surface wear fault is simulated by cutting a part of the tooth along the axis direction of the tooth. Pictures of the damaged sun gears are shown in Fig. 4.

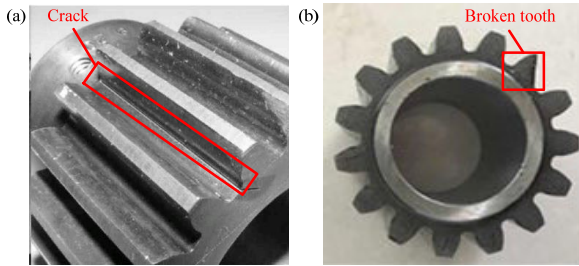


FIGURE 4. Sun gear damage: (a) crack and (b) wear.

It is well known that vibration-based analysis is an effective method for diagnosing mechanical system faults, the vibration signals are collected by tri-axial accelerometer sensors in our experiments, and the sampling rate is 20000 Hz. The sensors are mounted on the outer case of the planetary gearbox, as shown in Fig. 5. To simulate the general planetary gearbox operating conditions, two experiments are performed under the same loading conditions and two different motor speeds. Case 1 is the fault of a crack under a motor speed of 1500 r/min and a 900 N•m loading condition. Case 2 is the fault broken teeth under a motor speed of 1200 r/min and a 900 N•m loading condition. The corresponding characteristic frequencies of the planetary gearbox under two different motor speeds are summarized in Table 2.

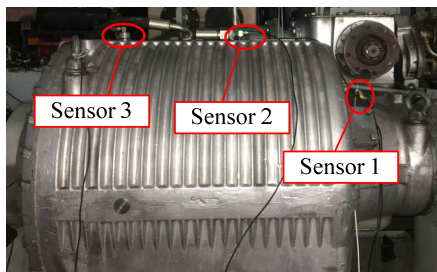


FIGURE 5. Layout of the sensor points.

TABLE 2. Motor speed and characteristics frequencies of the planetary gearbox.

Motor speed (rpm)	f_0 (Hz)	f_m (Hz)	f_{m1} (Hz)	f_{m2} (Hz)	f_{mK3} (Hz)	f_{s3} (Hz)
1200	20	360	580	260	423.52	14.12
1500	25	450	725	325	529.4	17.65

Note: f_0 is the rotating frequency of the motor. f_m is the meshing frequency. f_{m1} and f_{m2} are the meshing frequency of the main pump and return pump, respectively. f_{mK3} is the meshing frequency of the K3 row. f_{s3} is the fault characteristic frequency of the sun gear in the K3 row.

C. EXPERIMENTAL RESULTS AND DISCUSSION

In this section, we apply the proposed nonconvex WATV denoising method to diagnosis the faults of the planetary

gearbox as introduced in the part B. To further demonstrate the effectiveness of the proposed method, the analysis results are compared with those of the soft thresholding denoising method and the BPD method. Compared with the vibration signals of the vertical direction, the signals of the horizontal direction are more sensitive to the damage. Because the vertical vibrations are constrained by the gravity of the test rig and the basement, their vibration amplitudes are not as large as the horizontal ones. Therefore, the vibration signals of the horizontal direction are considered in this paper. Moreover, the faults are located on the sun gear in the K3 row, so the vibration signal of the measuring point 3 closest to the fault source is selected for analysis.

In this work, the wavelet threshold is

$$t_l = \sigma_l \sqrt{2 \ln(N_l)} \quad (31)$$

where N_l is the length of the level l detail signal of wavelet analysis, σ_l is the noise standard deviation of level l detail signal. And the regularization parameter is suggested using

$$\lambda_j = 2.5\eta\sigma / 2^{j/2}, \quad \beta = (1 - \eta) \sqrt{N} \sigma / 4 \quad (32)$$

where j is the wavelet scale, η is the relative weight control parameter with a nominal value of $\eta = 0.95$.

1) CASE 1

The testing result of case 1 is shown in Fig. 6 the time-domain waveform of the vibration signal with a duration of 2 s is illustrated in Fig. 6 (a). The fault signature cannot be identified because the useful periodic pulses are buried in the strong background noise. The frequency spectrum and the magnified spectrum of the signal are shown in Fig. 6 (b) and (d), respectively. The rotating frequency of the motor f_0 and the meshing frequency of the fixed-shaft gear f_m , as well as their multiples are predominant. Moreover, there are many sidebands equally spaced around the meshing frequency f_m and its multiples, and the sideband spacing is the rotating frequency f_0 . The main frequency components in the Hilbert envelope spectrum (shown in Fig. 6 (c)) are the rotating frequency f_0 and its double frequency. Meanwhile, the meshing frequency f_m and its sidebands are also prominent in Fig. 6 (c). However, the fault features of the planetary gearbox cannot be identified

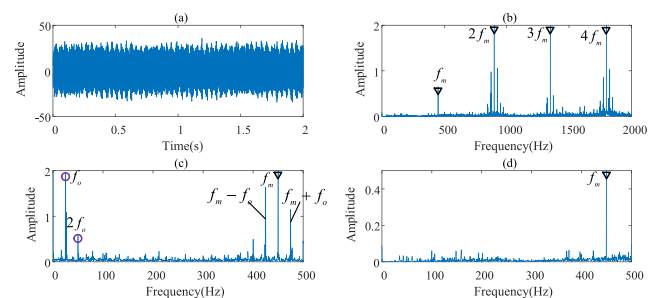


FIGURE 6. Original vibration signals of case 1: (a) the time-domain waveform; (b) frequency spectrum; (c) Hilbert envelope spectrum; (d) magnified spectrum.

from Fig. 6, because the useful information is buried in the strong background noise.

The meshing frequency of the main pump f_{m1} and the return pump f_{m2} cannot be identified in the spectrum. Because the measuring point 3 is the farthest point from the hydraulic station, so the vibration signals cannot be effectively collected by the vibration sensor of measuring point 3. Even the vibration signal can be collected, considering the complexity of the transmission path, those vibration characteristics are relatively weak in the collected signal, which cannot be identified. The meshing frequency of the K3 row f_{mK3} is also not identified because of the weak vibration characteristics.

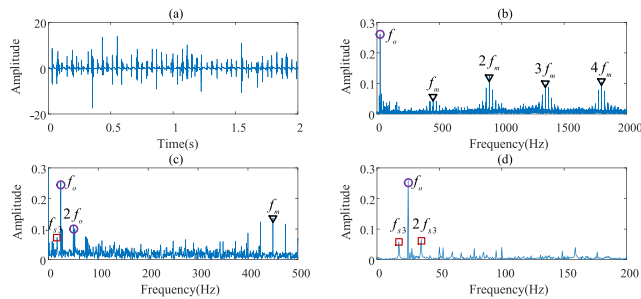


FIGURE 7. Analysis results of nonconvex WATV denoising in case 1: (a) the time-domain waveform; (b) frequency spectrum; (c) Hilbert envelope spectrum; (d) magnified spectrum.

To extract the useful fault information, the proposed method is utilized to process the vibration signals. The results are shown in Fig. 7. From the time domain waveform of Fig. 7 (a), many irrelevant interference components and noises have been removed and strong periodic impulses are clearly revealed. From the frequency and the magnified spectrum of Fig. 7 (b) and (d), respectively, it is obvious that except for the meshing frequency f_m and its multiples, as well as the sidebands with an equal interval of the rotating frequency f_0 , the gear fault characteristic frequency f_{s3} and its double frequency are successfully extracted in the spectrums. The fault characteristic frequency f_{s3} is also clearly revealed in the Hilbert envelope spectrum, as shown in Fig. 7 (c). Thus, the proposed nonconvex WATV method successfully detects the crack fault of the planetary gearbox. More specifically, the fault features of the planetary gearbox are successfully extracted utilizing the proposed method.

For comparison, we also use the soft threshold denoising method to process the same vibration signal, and the results are shown in Fig. 8. Although periodic pulses exist in the time-domain waveform as shown in Fig. 8 (a), they are not as obvious as those in the time-domain waveform as shown in Fig. 7 (a), which indicates that many noise and irrelevant interference components still exist in the analysis results. In both frequency spectrum and the Hilbert envelope spectrum, only the meshing frequency f_m , rotating frequency f_0 and their multiples can be obtained; the fault characteristic frequency used to monitor the health status of the gearbox cannot be observed from Fig. 8 (b), (c), or (d).

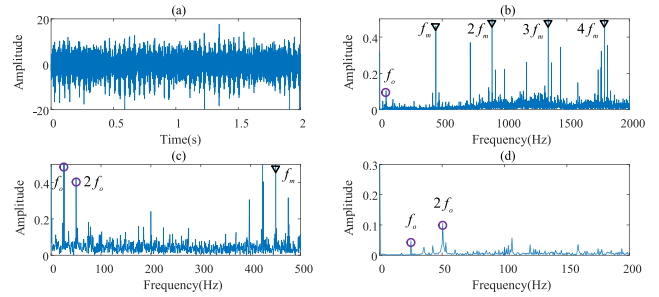


FIGURE 8. Analysis results of soft thresholding denoising in case 1: (a) the time-domain waveform; (b) frequency spectrum; (c) Hilbert envelope spectrum; (d) magnified spectrum.

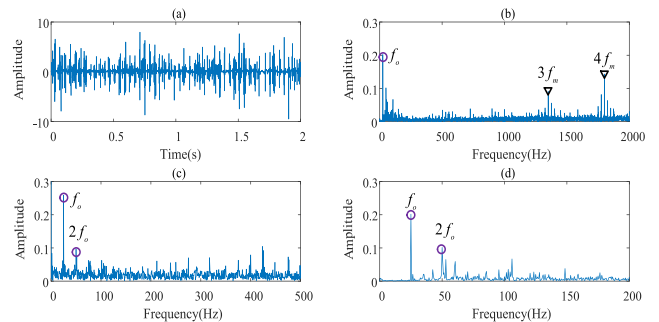


FIGURE 9. Analysis results of BPD in case 1: (a) the time-domain waveform; (b) frequency spectrum; (c) Hilbert envelope spectrum; (d) magnified spectrum.

For further comparison, the BPD denoising method is sequentially applied to the same vibration signals, with the same parameters as those used in the proposed method. The results are shown in Fig. 9. From the time domain waveform of Fig. 9 (a), the estimated periodic transient components are not as accurate as those estimated by the proposed method, because the amplitudes of the transients are underestimated. Moreover, the analysis results are not sparse enough and there are still many interference components, which effect the fault diagnosis. Thus, no useful fault characteristic frequency can be extracted from the spectrums, as shown in Fig. 9 (b), (c), and (d).

According to the mathematical principle, the analysis results of the soft threshold and the BPD method should be the same. However, compared Fig. 8 and Fig. 9, we find that the results are not the same. This is mainly because the soft threshold method is employed to deal with the decomposition coefficient after wavelet decomposition, and then reconstruct the signal using the estimated wavelet coefficients, so as to realize the noise reduction of the measured signal. The BPD method is directly to analyze the original measured signal. In addition, the parameter settings of the two methods are different. In soft threshold denoising, the threshold is set by equation (31), while in BPD algorithm, the regularization parameter is determined by equation (32). Therefore, the results of the two methods are different.

2) CASE 2

The waveform of the vibration signal for case 2 and its corresponding spectrums are shown in Fig. 10. As seen from

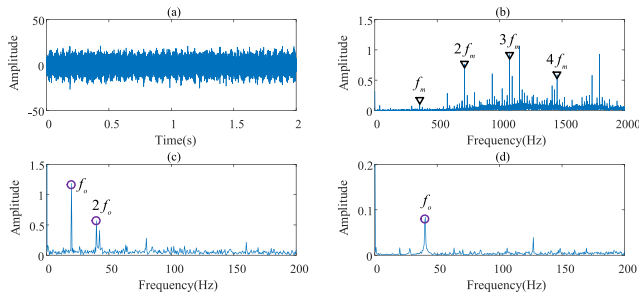


FIGURE 10. Original vibration signals of case 2: (a) the time-domain waveform; (b) frequency spectrum; (c) Hilbert envelope spectrum; (d) magnified spectrum.

Fig. 10 (b), (c), and (d), the main frequency components in the frequency spectrum and the Hilbert envelope spectrum are the meshing frequency and the rotating frequency, as well as their multiples. Neither the fault characteristic frequency nor its multiples can be identified in Fig. 10.

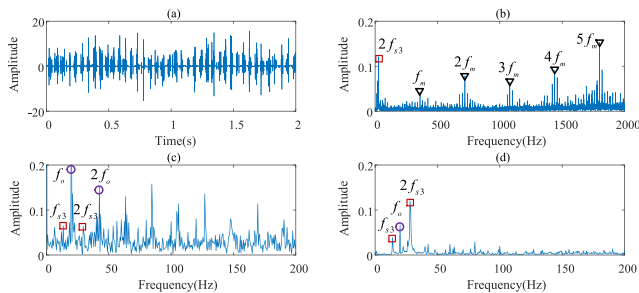


FIGURE 11. Analysis results of nonconvex WATV denoising in case 2: (a) the time-domain waveform; (b) frequency spectrum; (c) Hilbert envelope spectrum; (d) magnified spectrum.

Then, the proposed nonconvex WATV denoising method is used to process the vibration signal in Fig. 10 (a), and the results are shown in Fig. 11. In the time-domain waveform in Fig. 11 (a), the transients in the analysis results are apparent, and the period of the transients is obvious. From the spectrums of Fig. 11 (b), (c), and (d), it is noted that the dominant frequencies are not only the meshing frequency f_m and the rotating frequency f_0 , but also their multiples and sidebands; the fault characteristic frequency f_{s3} and its multiple also clearly appear in the Hilbert envelope spectrum (shown in Fig. 11 (c)) and the magnified spectrum (shown in Fig. 11 (d)). Based on the above-mentioned analysis, we can conclude that there is a localized fault on the sun gear.

For further comparison, both the soft thresholding denoising method and the BPD method are employed to analyze the vibration signals, and the results are shown in Fig. 12 and Fig. 13, respectively. Obviously, there are still many noise and interference components in the time-domain waveform of the results of the two methods, which results in the periodic characteristics of the estimated transient components in Fig. 12 (a) and Fig. 13 (a) being less prominent than those in Fig. 11 (a). In the spectrums shown in Fig. 12, in addition to the main frequency components

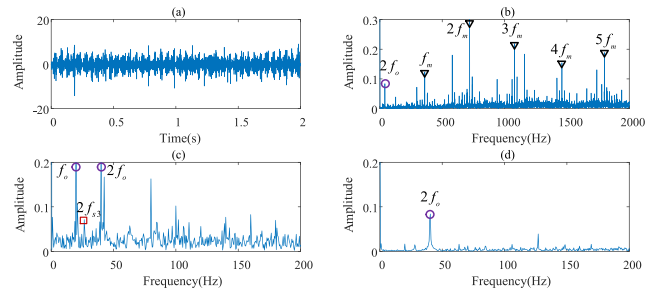


FIGURE 12. Analysis results of soft thresholding denoising in case 2: (a) the time-domain waveform; (b) frequency spectrum; (c) Hilbert envelope spectrum; (d) magnified spectrum.

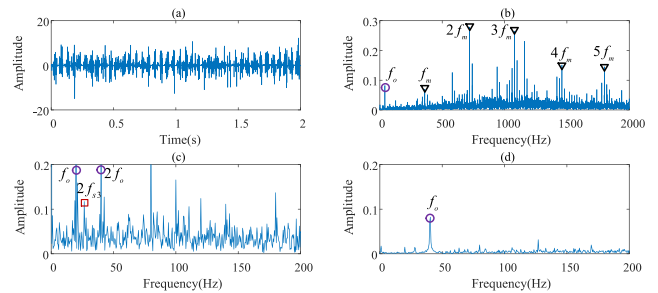


FIGURE 13. Analysis results of BPD in case 2: (a) the time-domain waveform; (b) frequency spectrum; (c) Hilbert envelope spectrum; (d) magnified spectrum.

as shown in Fig. 10, there is also a harmonic component of the fault characteristic frequency $2f_{s3}$ in the Hilbert envelope spectrum shown in Fig. 12 (c). Despite the existence of the harmonic frequency, other evidence is necessary for the definite diagnosis of a sun gear fault. However, no other useful features related to the sun gear can be extracted from the analysis results. Similarly, in the results of the BPD method, only the second-harmonic component of the fault characteristic frequency $2f_{s3}$ exists in the Hilbert envelope spectrum shown in Fig. 12 (c), and again, no more useful features related to the fault can be extracted. Compared with the results of the proposed method shown in Fig. 11, neither the soft thresholding denoising method nor the BPD method provide satisfactory performance in the fault diagnosis of the planetary gearbox.

V. CONCLUSION

This paper proposes a nonconvex WATV denoising method for the purpose of detecting faults in a planetary gearbox. The propose method formulates wavelet thresholding and TV as a unified problem. We use a TV minimization algorithm to reconstruct the retained wavelet coefficients, so the pseudo-Gibbs oscillations phenomena caused by pure wavelet thresholding are removed. In order to strongly induce the wavelet sparsity, we consider a modification of the TV optimization problem where the L1 norm regularizer is replaced by a nonconvex penalty function. However, the use of a nonconvex regularizer converts the denoising problem into a nonconvex optimization problem. Thus, the conditions on

the nonconvex degree parameter of the nonconvex penalties are addressed to ensure that the total objective function is strictly convex. Then, the convex optimization solver SALSA is introduced to find the global optimal solution.

A planetary gearbox test rig is established and two modes of faults are simulated. The vibration signals obtained under two different motor speeds are used to verify the effectiveness of the proposed method in the diagnosis of localized faults in a sun gear. Comparisons to the soft thresholding denoising and BPD methods demonstrate that the proposed method can better preserve feature components of interest and can significantly improve the estimation accuracy, thus providing improved fault detection accuracy. The proposed nonconvex WATV denoising method is concluded to be an effective tool for fault diagnosis of a planetary gearbox.

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