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Adaptive Nonsingular Terminal Sliding Model Control for Permanent Magnet Synchronous Motor Based on Disturbance Observer

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ABSTRACT In order to shorten the response time and improve robustness for a permanent magnet synchronous motor (PMSM) control system, an adaptive nonsingular terminal sliding model control (NTSMC) equipped with a disturbance observer is presented for PMSM. First, an improved exponential reaching law is adopted to adaptively adjust exponential and constant approach speed in the adaptive NTSMC. Second, the disturbance observer is devised to observe the load torque and external disturbance and feeds back them to the adaptive NTSMC to compensate. Simulation and experimental results applied to PMSM show that the proposed approach can quickly track load torque and external disturbance and the system has advantages of small overshoot, static error, rapidity, and higher robustness.

INDEX TERMS PMSM, adaptive NTSMC, disturbance observer.

I. INTRODUCTION

PMSM (Permanent Magnet Synchronous Motor) has characteristics of simple structure, small size, less weight, high efficiency, high factor, rotor heat out of question, better over loading, lesser moment of inertia and torque ripple. It has been widely used in the fields of numerical control machine tools, medical instruments, instruments and meters, aviation and spaceflight. It is well known that traditional control schemes, such as PID (proportion integral derivative), are already widely used in the PMSM control system due to its simple implementation [1]. However, the PMSM system is the nonlinear, time-varying and complex with unavoidable and unmeasured disturbances as well as parameter variations. It is very difficult to achieve a satisfactory performance in the entire operating range only by using traditional control algorithm [2]–[5].

In recent years, various modern control techniques have been presented to improve the different aspects performance of the PMSM control system [6]–[12]. Among these methods, the sliding mode control (SMC) method is considered to be efficient to increase the disturbance rejection and the robustness property of the PMSM system and have been improved in intensive studies [13]–[16]. To further improve

the dynamic performance of the SMC, a direct method is to introduce the nonlinear sliding surfaces. Terminal sliding mode (TSM), as one of nonlinear sliding surfaces, can ensure the finite-time convergence [17], [18]. The TSM control method with the nonlinear term can guarantee system converge on a specified trajectory in finite time. In order to solve singular problem, [19] presented the nonsingular terminal sliding model control. Reference [20] adopted new reaching law to design control law. This method not only improved system response but also reduced chattering effectively. Reference [21] designed variable-rated exponential reaching law, but the method didn't consider the selection of reaching law when state variable is close to zero.

In our previous study [22], the L_1 norm of state variable was used to design adaptive variable-rated exponential reaching law. However, the problems of system uncertainty and external disturbance weren't considered. In this paper, based on [22] and [23], an improved exponential reaching law is present to design control law. In order to shorten the reaching time and weaken system chattering, exponential and constant approach speed can adaptively adjust to the equilibrium position in accordance with the distance of state variables. In additional, in terms of system uncertainty,

external disturbance and load torque, according to the basic idea of disturbance observer which can estimate disturbances and feedback them to foreword channel immediately to compensate [24], [25]. This paper designs the disturbance observer (DOB) on basis of the theory of Luenberger state observer instead of taking the boundary value. The proposed disturbance observer can observe the values of load torque, system uncertainty and external disturbance in real time, and then feed back them to the sliding model controller. This method extracts the advantages of the adaptive NTSMC and the disturbance observer, which not only shortens the reaching time and weakens system chattering, but also improves system robustness. The application of the proposed approach in a simulation and engineering system indicate its validity.

II. MATHEMATICAL MODEL OF PMSM

In order to simplify analysis, in this paper, the PMSM stator adopts three-phase symmetrical windings, and the rotor has a permanent magnet structure. Assuming that the magnetic field is sinusoidally distributed in space, without considering the effects of hysteresis and eddy current loss, take the d axis in line with the excitation axis of the permanent magnet, and the q axis follows the direction of rotation of the rotor and leads the d axis $\pi/2$ electrical angle. Mathematical model of magnetic synchronous motor in d - q coordinate system ($L_d = L_q = L$ for surface mounted PMSM) as follows [26].

$$\begin{cases} u_d = Ri_d - \omega_r L_q i_q + L_d \frac{di_d}{dt} \\ u_q = Ri_q + \omega_r L_d i_d + L_q \frac{di_q}{dt} + \omega_r \psi \end{cases} \quad (1)$$

where u_d, u_q, i_d and i_q are stator voltage and current components in the d - q coordinate system respectively; L_d, L_q are respectively inductances on d axis and q axis; R points motor winding resistance; ω_r indicates the electric angular velocity; and ψ presents the flux of permanent magnet.

The torque equation of PMSM is:

$$T_e = 1.5p[\psi i_q + (L_d - L_q)i_d i_q] \quad (2)$$

where T_e is the electromagnetic torque; and p refers to pole pairs. For the surface PMSM, $L_d = L_q = L$, thus the torque equation is reduced to as follows:

$$T_e = 1.5p\psi i_q \quad (3)$$

The motion equation of PMSM is:

$$T_e - T_L - F\omega = J \frac{d\omega}{dt} \quad (4)$$

where T_L is load torque; ω points the mechanical angular velocity of the motor; F means friction coefficient of the rotor and load; and J implies inertia moment.

The following equation considers the uncertainty of the PMSM system.

$$\begin{aligned} \frac{d\omega}{dt} &= \left(\frac{1.5p\psi}{J} + \Delta a \right) i_q - \left(\frac{F}{J} + \Delta b \right) \omega - \left(\frac{1}{J} T_L + \Delta d \right) \\ &= \frac{1.5p\psi}{J} i_q - \frac{F}{J} \omega - \frac{T_L - J i_q \Delta a + J \omega \Delta b + J \Delta d}{J} \\ &= \frac{1.5p\psi}{J} i_q - \frac{F}{J} \omega - \frac{g(t)}{J} \end{aligned} \quad (5)$$

where $g(t) = T_L - J i_q \Delta a + J \omega \Delta b + J \Delta d$.

III. DESIGN OF THE ADAPTIVE NTSMC

A. SLIDING SURFACE

$$\begin{cases} x_1 = \omega^* - \omega \\ x_2 = \dot{x}_1 = -\dot{\omega} \end{cases} \quad (6)$$

Equation (7) is obtained from equation (5) and equation (6):

$$\begin{aligned} \dot{x}_2 &= -\ddot{\omega} \\ &= -\frac{F}{J} x_2 - \frac{1.5p\psi}{J} \dot{i}_q + \frac{\dot{g}(t)}{J} \end{aligned} \quad (7)$$

Assuming that $g(t)$ is the total uncertainty, thus there is the following equation:

$$g(t) = T_L - J i_q \Delta a + J \omega \Delta b + J \Delta d \quad (8)$$

From equation (6), (7) and (8), the speed error state equations can be presented as follows:

$$\begin{cases} \dot{x}_1 = x_2 \\ \dot{x}_2 = -\frac{1.5p\psi}{J} \dot{i}_q - \frac{F}{J} x_2 + \frac{\dot{g}(t)}{J} \end{cases} \quad (9)$$

For system (9), in order to achieve excellent performance, the nonsingular terminal sliding surface is designed as follows:

$$s = x_1 + \frac{1}{\beta} x_2^{p/q} \quad (10)$$

where $\beta > 0$, p and q are positive odd integers under the situation of $1 < p/q < 2$.

B. CONTROL LAW

For system (9), the ordinary exponential reaching law is adopted to design the SMC. The reaching law is as follow:

$$\dot{s} = -\varepsilon \text{sgn}(s) - ks, \quad \varepsilon > 0 \quad (11)$$

The reaching law includes the index reaching item $-ks$, and the constant rate reaching item $-\varepsilon \text{sgn}(s)$. In equation (11), k and ε are constant and not self-adaptive, so system convergence characteristics cannot achieve the best performance for different position state variables. In here, an adaptive variable rate exponential reaching law is present by the following:

$$\dot{s} = -\varepsilon \frac{1}{1 + c \|x\|_1} \text{sgn}(s) - (k + c \|x\|_1) s \quad (12)$$

where, $k > 0$, $\varepsilon > 0$, $c > 0$, and $\|x\|_1 = \sum_{i=1}^n |x_i|$ is the first-order norm of state variables of the system, $n > 0$.

The adaptive variable-rated exponential reaching law (12) introduced the first-order norm of state variables can adaptively adjust its exponential approach speed and constant approach speed. The instant result of exponential reaching item is $s = s(0)e^{-(k+c\|x\|_1)t}$ for equation (12). When $\|x\|_1$ is large and the exponentially decaying speed is much greater than that in equation (11), the reaching time can be greatly shortened. Meantime, the reaching rate $\varepsilon \frac{1}{1+c\|x\|_1}$ of constant rate reaching item is far less than ε of equation (11). When $\|x\|_1$ is small, by increasing the adjustment coefficient c , the reaching time of the sliding mode can be shortened and system chattering can be weakened. When the selected state variable x converges to zero, the adaptive variable-rated exponential reaching law will decrease to the constant index reaching law.

In the consideration of the speed error state equation (9), the sliding surface equation (10) and the reaching law (12), the PMSM control law can be designed as follows:

$$i_q = \frac{J}{1.5p\psi} \left\{ \int_0^t \left[-\frac{F}{J}x_2 + \beta \frac{q}{p} x_2^{2-\frac{p}{q}} + \varepsilon \frac{1}{1+c\|x\|_1} \text{sgn}(s) + (k+c\|x\|_1)s \right] dt + \frac{g(t)}{J} \right\} \quad (13)$$

C. STABILITY ANALYSIS

Choosing Lyapunov function:

$$V = \frac{1}{2}s^2 \quad (14)$$

and taking the derivative of sliding surface (10) along system (9):

$$\dot{s} = \dot{x}_1 + \frac{1}{\beta} \frac{p}{q} x_2^{\frac{p}{q}-1} \dot{x}_2 = x_2 + \frac{1}{\beta} \frac{p}{q} x_2^{\frac{p}{q}-1} \dot{x}_2 \quad (15)$$

Then

$$\begin{aligned} \dot{V} &= s\dot{s} = s \left(x_2 + \frac{1}{\beta} \frac{p}{q} x_2^{\frac{p}{q}-1} \dot{x}_2 \right) \\ &= \frac{1}{\beta} \frac{p}{q} x_2^{\frac{p}{q}-1} \left\{ s [g(t) - l_g \text{sgn}(s)] - \varepsilon \frac{1}{1+c\|x_1\|} |s| - (k+cx_1)s^2 \right\} \quad (16) \end{aligned}$$

Two different cases will be discussed as follows:

(1) When $x_2 \neq 0$

$$\dot{V} = s\dot{s} \leq \frac{1}{\beta} \frac{p}{q} x_2^{\frac{p}{q}-1} \left\{ -\varepsilon \frac{1}{1+c\|x_1\|} |s| - (k+cx_1)s^2 \right\} \leq 0 \quad (17)$$

In this case, the system will converge to zero in finite time.

(2) When $x_2 = 0$

$$\dot{x}_2 = g(t) - l_g \text{sgn}(s) - \varepsilon \frac{1}{1+c\|x_1\|} |s| - (k+cx_1)s \quad (18)$$

Then

$$\begin{cases} \dot{x}_2 \leq -\varepsilon \frac{1}{1+c\|x_1\|}, & s > 0 \\ \dot{x}_2 \geq \varepsilon \frac{1}{1+c\|x_1\|}, & s < 0 \end{cases} \quad (19)$$

It can be seen from equations in (18) that, when $s > 0$ and $x_2 = 0$, x_2 is not in steady state and will change into $x_2 < 0$ in finite time. Besides, the sliding mode surface will be achieved in finite time, which can be proved in the case of $x_2 \neq 0$. Similarly, when $s < 0$ and $x_2 = 0$, x_2 is not in steady state and will change into $x_2 > 0$ in finite time. At the same time, the system will reach the sliding mode surface from initial state in finite time.

The results show that in a limited time, the sliding mode can be obtained anywhere in the phase plane.

IV. DISTURBANCE OBSERVER

A. DESIGN OF DISTURBANCE OBSERVER

It can be seen from the control law (13) that $g(t) = T_L - J i_q \Delta a + J \omega \Delta b + J \Delta d$ is an unknown quantity and cannot be measured. In this paper, the disturbance observer is designed to observe the values of disturbance and load torque in real time.

ω can be measured and $\frac{d\omega}{dt} = \frac{1.5p\psi}{J} i_q - \frac{F}{J} \omega - \frac{g(t)}{J}$. The disturbance observer can be designed by the theory of Luenberger linear observer.

The PMSM speed system can be presented as follows:

$$\begin{cases} \dot{x} = Ax + Bu \\ y = Cx \end{cases} \quad (20)$$

where state variable $x = [\omega \ g(t)]^T$, output variable $y = \omega$, input variable $u = 1.5p\psi i_q$, $A = \begin{bmatrix} -\frac{F}{J} & -\frac{1}{J} \\ 0 & 0 \end{bmatrix}$, $B = [\frac{1}{J} \ 0]^T$ and $C = [1 \ 0]$.

It can be known from the modern control theory that (A, C) is observable, thus the state observer exists.

Constructing the state observer for system (20)

$$\begin{cases} \dot{\hat{x}} = A\hat{x} + Bu + L(y - \hat{y}) \\ \hat{y} = C\hat{x} \end{cases} \quad (21)$$

where $\hat{x} = [\hat{\omega} \ \hat{g}(t)]$ is the estimated value of x , \hat{y} is the estimated value of y , and $L = [l_1 \ l_2]$ points the feedback matrix.

The error equation of this observer is as the following:

$$\dot{e} = \dot{x} - \dot{\hat{x}} = (A - LC)e \quad (22)$$

For equation (22), if the eigenvalues of matrix $(A - LC)$ all have negative real parts, the state error e will converge to zero asymptotically. The poles of equation (22) should be placed in the left half plane through pole assignment to ensure the system error converging to zero.

Substituting $\hat{g}(t)$ into equation (13) obtains the following equation:

$$i_q = \frac{J}{1.5p\psi} \left\{ \int_0^t \left[-\frac{F}{J}x_2 + \beta \frac{q}{p} x_2^{2-\frac{p}{q}} + \varepsilon \frac{1}{1+c\|x\|_1} \text{sgn}(s) + (k+c\|x\|_1)s \right] dt + \frac{\hat{g}(t)}{J} \right\} \quad (23)$$

B. SIMULATION

In order to verify whether the disturbance observer can track the change of load and external disturbance, some simulated experiments are carried out. According to [22], the parameters of the adaptive NTSMC are $\beta = 1000$, $p = 9$, $q = 7$, $k = 45$, $\varepsilon = 80$ and $c = 50$. Fig.1 and Fig.2 show the results.

It can be seen from Fig.1 and Fig.2 that, when $g(t) = -5$, $g(t)$ estimation can make response to changes quickly and the error between $g(t)$ and $g(t)$ estimation is close to zero. When $g(t) = 0.05 \sin(t)$, $g(t)$ estimation can track it quickly and exactly.

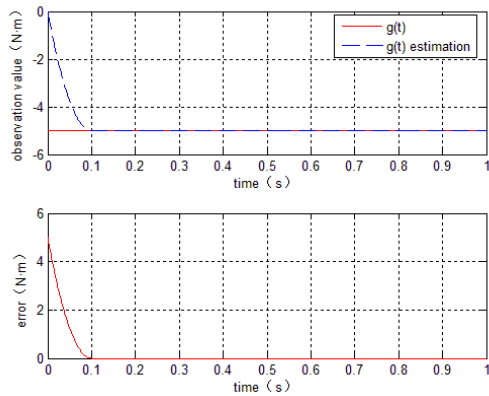


FIGURE 1. Fixed load observation and error.

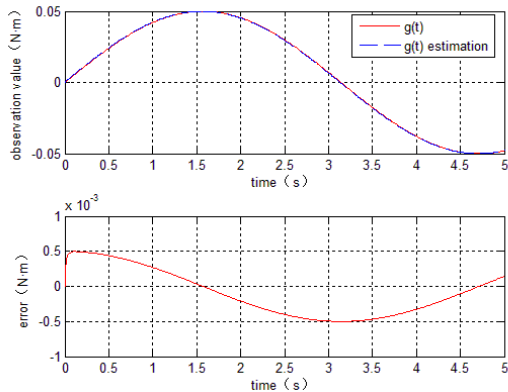


FIGURE 2. Dynamic load observation and error.

V. EXPERIMENTAL ANALYSIS

To verify the effectiveness of the designed controller, there is a hardware control system based on TMS320F28335 DSP for PMSM. The field-oriented control of $i_d = 0$ is adopted in this system. Fig.3 is the PMSM speed control system structure. Fig.4 shows the PMSM experimental platform. Specific parameters of PMSM are as follows: $L_d = L_q = 6.55mH$, $R = 0.901\Omega$, $\psi = 0.031Wb$, $p = 4$ and $J = 1.2 \times 10^{-4}kg \cdot m^2$. SVPWM is included in the whole algorithm, which is carried out in the program of the fixed point DSP TMS320F28335 under 100 MHz clock frequency. The utilization of C-program applies the control algorithm.

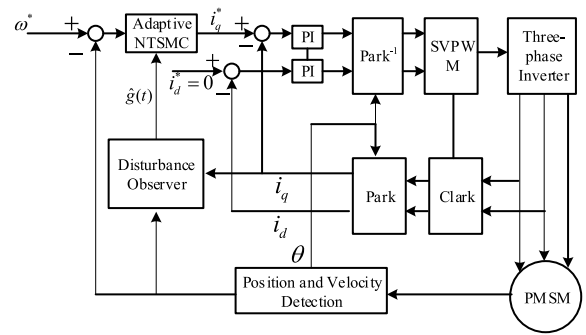


FIGURE 3. The PMSM speed control system structure.

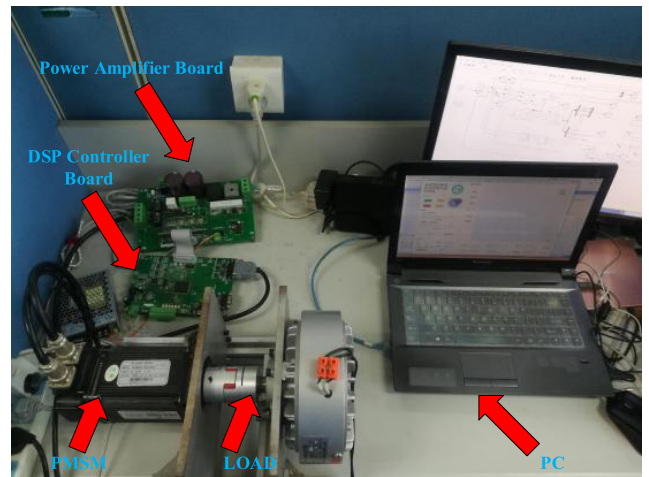


FIGURE 4. PMSM experimental platform.

The q axis reference current's saturation limit is $\pm 8A$. A three-phase voltage source PWM inverter is used to drive PMSM. This takes the advantage of IPM (intelligent power module) PS21865. The Hall-effect device is used to measure the phase currents. Meanwhile, the currents then are converted through the two 12-bit A/D converters. A SK-1A controller and a CZF25-eddy current brake are included in the AC speed regulation system to produce the braking torque, which can be applied as the external load disturbance. Afterwards, the tuning excitation current can be used to regulate the output torque of eddy current brake. Here, excitation current is set as 0.5A with external torque equal to rated load. The control gains of both current-loops are set as $k_p = 3$ and $k_i = 5$. The parameters of the adaptive NTSMC are the same as those in simulation.

With the help of dual system, the feedback matrix L of disturbance observer can be obtained: $L = [-7 \ 12J - J^2]$. Fig.5 shows the effect of disturbance observer for torque when the given torque value changes from 0 to $3.2 N \cdot m$. The experimental result shows that the proposed disturbance observer can track the actual value precisely in real time.

Fig.6 shows that the single phase current of the PMSM without load at speed of 1200r/min under the control of adaptive NTSMC with DOB. Additionally, Fig.7 shows the

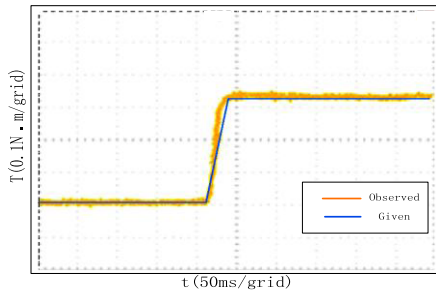


FIGURE 5. Observed value of disturbance observer.

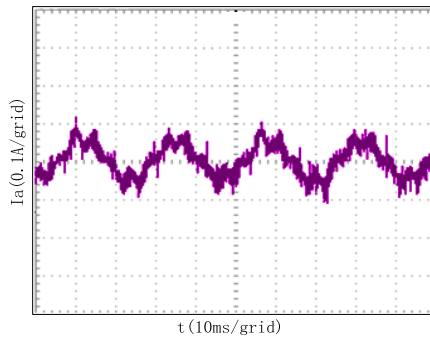


FIGURE 6. Current without load.

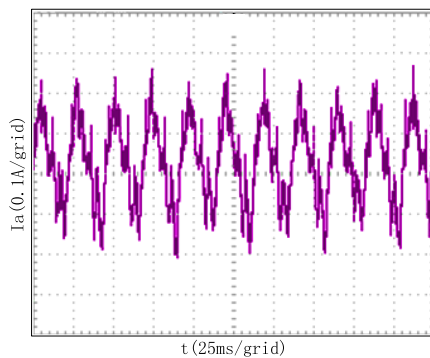


FIGURE 7. Current with load.

single phase current of the PMSM with the load of $3.2N \cdot m$. It can be seen the current presents a sine wave and is relatively stable.

Fig.8 shows the speed step response curve under the adaptive NTSMC with DOB and the NTSMC control. It can be seen clearly from Fig.8 (a) that, when PMSM speed is set as 1200r/min, the adaptive NTSMC with DOB response is 68.85%, faster than that of NTSMC control with only 2.3% overshoot. Fig.8 (b) shows that, when speed is set as 600r/min, the adaptive NTSMC with DOB response is 68.75% faster with smaller overshoot of just 2.17%. Fig.8 (c) shows that the adaptive NTSMC with DOB response is 42.86% faster with smaller overshoot of just 0.42% when speed is set as 300r/min. It can be concluded from these experimental results that the adaptive NTSMC with DOB can deliver faster speed response during start-up under the

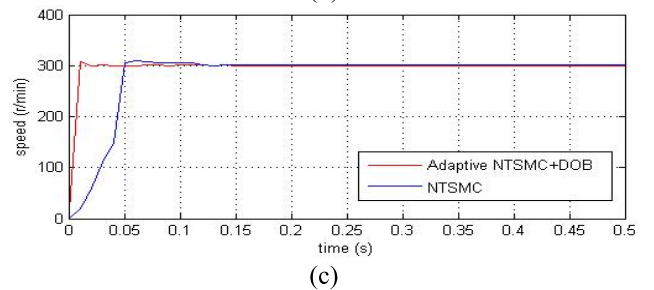
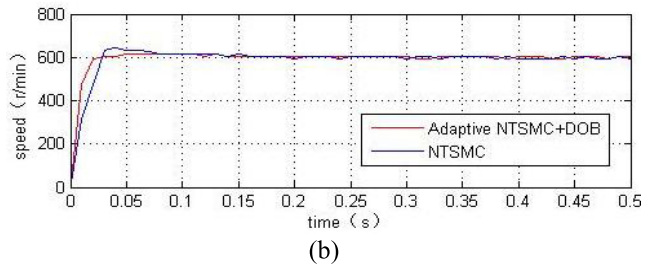
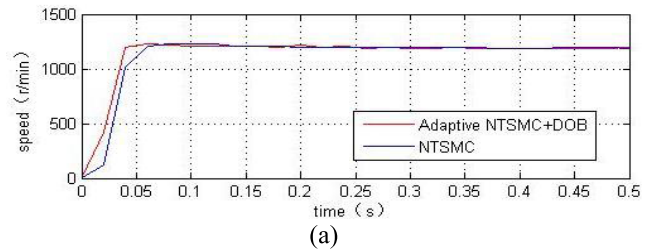


FIGURE 8. Speed step responses under the control of the adaptive NTSMC with DOB and NTSMC (a) 1200r/min. (b) 600r/min. (c) 300r/min.

comparison with NTSMC. Fig.9 shows the torque response without adding load when the target speed is 1200, 600, 300 r/min respectively. It can be clearly seen from figures that adaptive NTSMC with DOB has a fast torque response with smaller overshoot and chattering.

Fig.10 shows the speed response curve with a sudden change of speed from 1200r/min to 300r/min under the two controllers. It can be seen clearly that the adaptive NTSMC with DOB has less overshoot of only 4.33% and better velocity response, because it just uses 3ms to drop to 300r/min, but is 40% faster than NTSMC. Fig.11 displays the speed response when an abrupt load is added in case of the two controllers. When load abruptly changes, the adaptive NTSMC with DOB shows better robustness. Speed fluctuation of the control system under the NTSM with DOB technique is smaller, and the recovering time against disturbance is shorter. Fig.12 shows the response of the current when a sudden load is added. Here, the excitation current is set as 0.5A with external torque equal to rated load. It can be seen clearly that the adaptive NTSM with DOB method is able to track currents well with less time, smaller overshoot and better robustness

In order to better illustrate the superiority of the proposed algorithm, this paper introduces some control system performance indexes, such as overshoot (OS%), settling

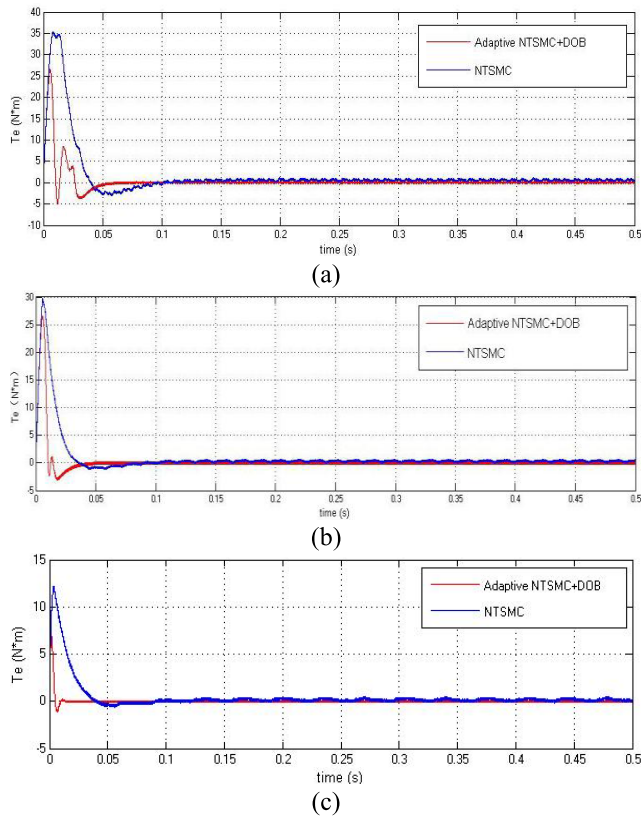


FIGURE 9. Torque responses under the control of the adaptive NTSMC with DOB and NTSMC (a) 1200r/min (b) 600r/min (c) 300r/min.

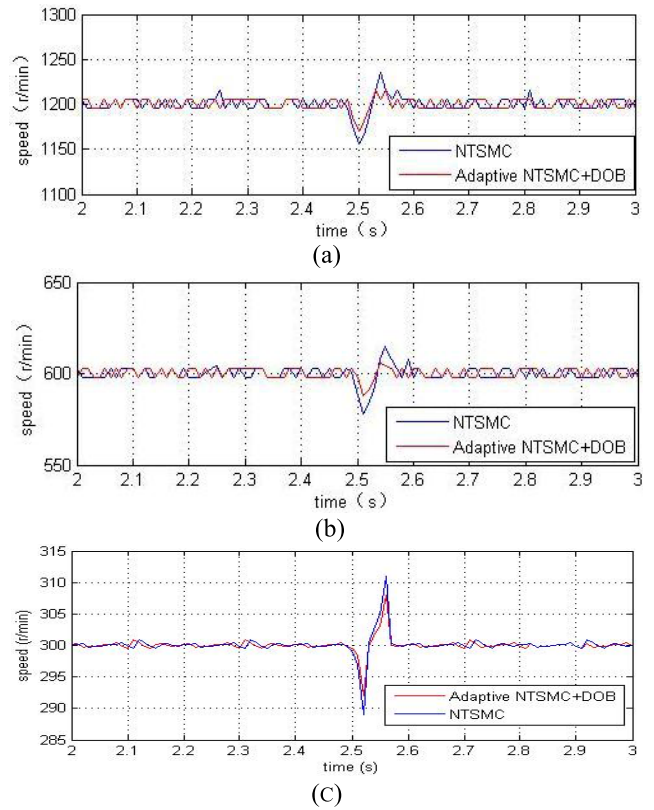


FIGURE 11. Speed responses of a sudden load under the control of the adaptive NTSMC with DOB and NTSMC (a) 1200r/min (b) 600r/min (c) 300r/min.

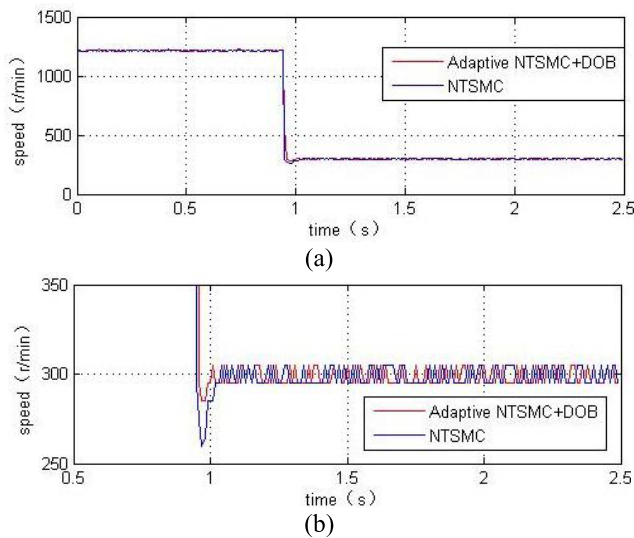


FIGURE 10. Speed responses with a sudden change of speed from 1200r/min to 300r/min under the control of the adaptive NTSMC with DOB and NTSMC (a) Speed responses curve (b) Local curve zoom.

time (t_s) and maximum decrease of speed caused by load torque. TABLE 1 demonstrates the comparison of performance indexes from the two methods under different conditions. It can be observed accurately that in all three aspects the adaptive NTSMC with DOB has a better performance

TABLE 1. Comparison of the major performance indices.

Control scheme	Index	OS (%)	t_s (ms)	Speed decrease/ (rpm)
300r/min	NTSMC	3.12	61	11
	adaptive NTSMC+DOB	2.3	19	8
600r/min	NTSMC	5.33	80	21
	adaptive NTSMC+DOB	2.17	25	12
1200r/min	NTSMC	1.33	70	46
	adaptive NTSMC+DOB	0.42	40	30

than NTSMC, and its response time is 42.86%, 68.75% and 68.85% respectively at 300r/min, and 600 r/min and 1200r/min, faster than NTSM. In addition, its overshoot is smaller. As shown in TABLE 2, with speed changing from 1200r/min to 300r/min, the adaptive NTSMC with DOB precisely tracks it with less time.

The experimental results show that the design of the new type of disturbance observer based terminal sliding mode control, applied to PMSM vector control speed regulation system is effective and feasible. It can not only well ensure the dynamic performance of the PMSM speed control system, but

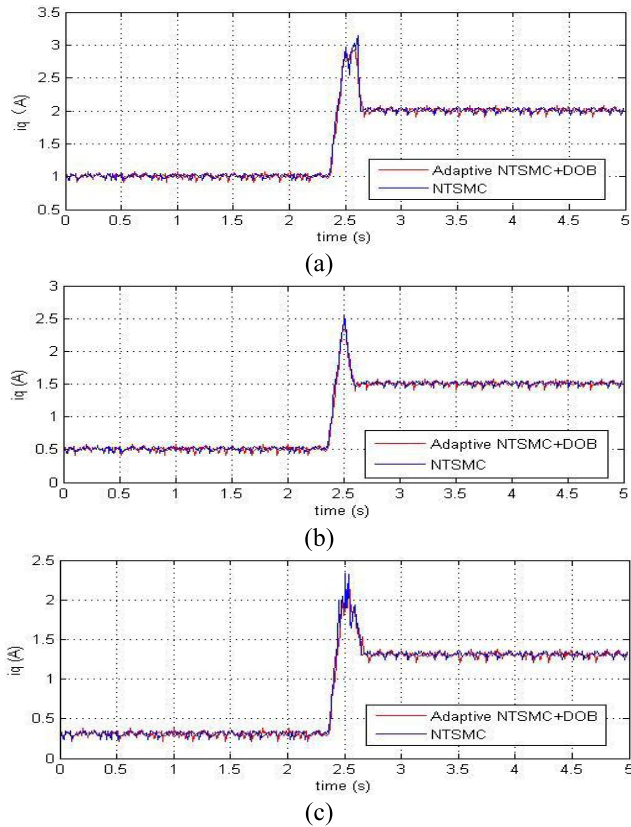


FIGURE 12. Current responses of a sudden load under the control of the adaptive NTSMC with DOB and NTSMC (a) 1200r/min (b) 600r/min (c) 300r/min.

TABLE 2. Speed change.

Control scheme		Index	OS (%)	t/(ms)
		1200r/min to 300r/min	NTSMC	14.67
	adaptive NTSMC+DOB	4.33	3	

also can effectively improve system robustness and weaken system chattering.

VI. CONCLUSIONS

In this paper, an adaptive nonsingular terminal sliding model control with disturbance observer is presented for PMSM system. The combination of the adaptive NTSMC and disturbance observer not only shortens system response time but also improves system robustness. The simulation and experimental results indicate that the proposed method has rapid and higher robustness, making lower chattering and improving system efficiency.

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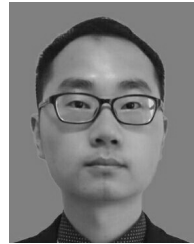
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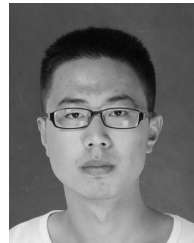
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