

## RESEARCH ARTICLE

# A Sensorless Control Method for Energy Recovery of EGTAC to Improve PEMFC Efficiency

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**ABSTRACT** The efficient operation of the air supply system, particularly the air compressor, is crucial in ensuring the performance of the proton exchange membrane fuel cells (PEMFCs). Nevertheless, its high parasitic power consumption is the main reason for the efficiency decline of the PEMFC. While exhaust gas energy recovery is a viable approach to enhance system efficiency, conventional exhaust gas recovery systems are not well-suited for PEMFC and the cathode exhaust gas is difficult to monitor in real-time. Therefore, this study proposes a sensorless method based on reinforcement learning for energy recovery of the exhaust gas turbine air compressor (EGTAC) in the PEMFC. Firstly, the air supply system with EGTAC and stack model are established. Subsequently, the relationship between the exhaust gas energy and the working performance of both the EGTAC and the PEMFC is elucidated. Additionally, a method based on a state observer is devised to estimate the characteristics of the exhaust gas in a PEMFC. Finally, compared to the model predictive control (MPC), this method enhances the EGTAC exhaust gas recovery rate by 19.1% and the fuel cell system efficiency by 3.7%. The optimization differs by a maximum of 6.5% compared to the control with sensors.

**INDEX TERMS** Exhaust gas energy recovery, estimation of exhaust gas characteristics, reinforcement learning, sensorless control.

## I. INTRODUCTION

In past few years, as the problems of energy scarcity and environmental degradation are becoming more and more apparent, countries all over the world have begun to search for new clean energy sources as substitutes for fossil energy sources and are actively exploring low carbon and high efficiency energy development routes. Hydrogen, as a secondary energy carrier, has been widely recognized and paid attention to because of its many advantages, like high efficiency, renewability, and stable energy conversion process [1], [2], [3]. PEMFCs, which utilizes hydrogen as its primary fuel, has increasingly gained attention and is seen as one of the most promising technologies to replace conventional internal combustion engines as the power source for vehicles

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in the future [4], [5], [6], [7]. The air supply system plays a crucial role in the PEMFC, in which the core component, the air compressor is in charge of supplying air to the fuel cell so that hydrogen and oxygen may react to produce energy and water. For higher power density and better performance, the PEMFC must operate at higher pressures, usually in the range of 2.0 to 2.5 atmospheres or so [8], [9]. Therefore, it is necessary for the air supply to promptly adapt to the change of load.

Some scholars used a variety of methods [10], [11], [12], [13], [14] to control and optimize the intake of the air supply system, which have achieved good results and improved the fuel cell's performance, but ignored the power loss of the air compressor. When the input pressure rises, the power of PEMFC increases, and the power consumption of air compressor will also be higher [15], [16], [17]. It is imperative that the air supply system's power consumption be

decreased, and that the overall efficiency of fuel cell system be increased. Several scholars have enhanced the efficiency of fuel cell systems by using energy management strategies [18], employing multi-objective optimization control strategies [19], and other methods. Recycling the system's exhaust energy is a mature method of increasing system efficiency [20]. Exhaust energy recovery has been a topic of active discussion in the research field of conventional engine vehicles and hybrid electric vehicles. This method improved the dynamic performance and economic performance of engine [21], [22].

Compared to the traditional engine vehicle, the fuel cell vehicles (FCVs) also have a large amount of available energy, such as the energy of the cathode outlet gas, the pressure energy in the hydrogen storage bottle and the waste heat energy of the radiator. Unused energy is discharged into the surrounding environment, accounting for nearly 50% of the total energy loss [23], [24]. Pei et al. [25] analyzed the effect of different connection forms of exhaust gas energy recovery device on the fuel cell system. In order to determine the best course and optimization direction for optimizing future twin-screw expanders for fuel cells, Wang et al. [26] investigated the operating characteristics of the expander installed at the cathode of the electric reactor under various working situations and geometries. Wei et al. [27] simulated the exhaust gas energy recovery process of fuel cells under different operating loads, and obtained the effects of inlet state, current density, etc. on the turbine recovery efficiency. It offers a foundation for researching the best ways to govern exhaust gas recovery. Santiago [28] et al proposed an optimization method for electrically assisted coupled turbine structure and designed a decoupling control method for air compressor and turbine to improve the turbine efficiency, however, the increase of electrical losses reduces the efficiency of the whole system. Li et al. [29] established a model of the exhaust gas recovery system of PEMFC by controlling the air excess rate to supply the air flow, and the outcomes of the simulation demonstrated that this technique may raise the system power by 13%. However, the turbine's power contributed to 18% of the total parasitic power consumption. Thus, while utilizing a turbine to recover energy can enhance the power performance and economy of PEMFC, it increases additional power consumption and requires a certain amount of time to reach operational status. Additionally, the turbine's response time is sluggish, which fails to meet the immediate gas supply demands of the fuel cell system [30], [31]. Therefore, there is growing interest in the fuel cell air compressor that can directly recover the cathode exhaust gas, necessitating more research and development of its exhaust gas recovery optimization control.

In addition, the above scholars' research on the cathode exhaust gas energy recovery system largely depends on the continuous monitoring of the cathode gas concentration to maintain the gas condition at the desired level. Nevertheless, the installation of a sensor at the cathode outlet of the

stack to measure cathode exhaust gas parameters is both expensive and difficult to install [32]. Additionally, the low temperature at the cathode outlet of the stack generates a significant quantity of water vapor, which diminishes the precision and responsiveness of the sensor [33]. The fuel cell is also susceptible to metal ions, leading to a diminished reliability of the sensor [34]. While the mass spectrometer can be employed for precise diagnosis and analysis of gas concentration, the closed structure of the PEMFC system necessitates modifications to the fuel cell stack. Specifically, this involves the insertion of a capillary gas sampling tube into the flow channel, resulting in increased complexity and cost. Furthermore, the mass spectrometer exhibits a notably prolonged response time to water vapor pressure. This, in turn, leads to dynamic alterations in the internal operational state of the fuel cell, ultimately yielding measurement findings that are deemed inaccurate [35], [36]. Thus, it is imperative to create an observer that relies on a precise model of the fuel cell system and observable signals. This will ensure that the predicted gas concentration aligns with the actual value and enable real-time estimation of exhaust characteristics [37]. Based on the above, the structure of the paper is illustrated in Figure 1 and the contributions of this paper are as follows:

(1) This study proposes an accurate estimation method of fuel cell cathode exhaust characteristics based on the observer to solve the problem that it is difficult to monitor the cathode outlet gas of the fuel cell system in real-time. The change of working characteristics of the air supply system and fuel cell system caused by the change of exhaust energy was explored.

(2) This study proposes a novel fuel cell air compressor that incorporates an exhaust gas recovery feature to efficiently recover and process the cathode exhaust gas from fuel cells, therefore streamlining the exhaust gas energy recovery system. A control method utilizing a reinforcement learning algorithm is developed to enhance the efficiency of exhaust gas recovery and fuel cell system, without the need for sensors.

## II. MODELING

This paper uses a mechanism modeling approach to establish the mathematical model of a PEMFC stack and air supply system. The assumptions are as follows: (1) It is assumed that the ideal gas law applies to all gases. (2) It is assumed that the internal temperature distribution of the stack is balanced and that the gas temperature of the electrode and the stack are equal. (3) It is assumed that the hydrogen flow is sufficient and the hydrogen pressure meets the requirements of the following air pressure.

### A. FUEL CELL STACK MODEL

Through the principle of electrochemistry, the stack model shows the output performance of the battery, and uses the law of conservation of matter to show the dynamic change of gas mass flow in the cathode flow field. By means of empirical and physical formulas, such as concentration polarization

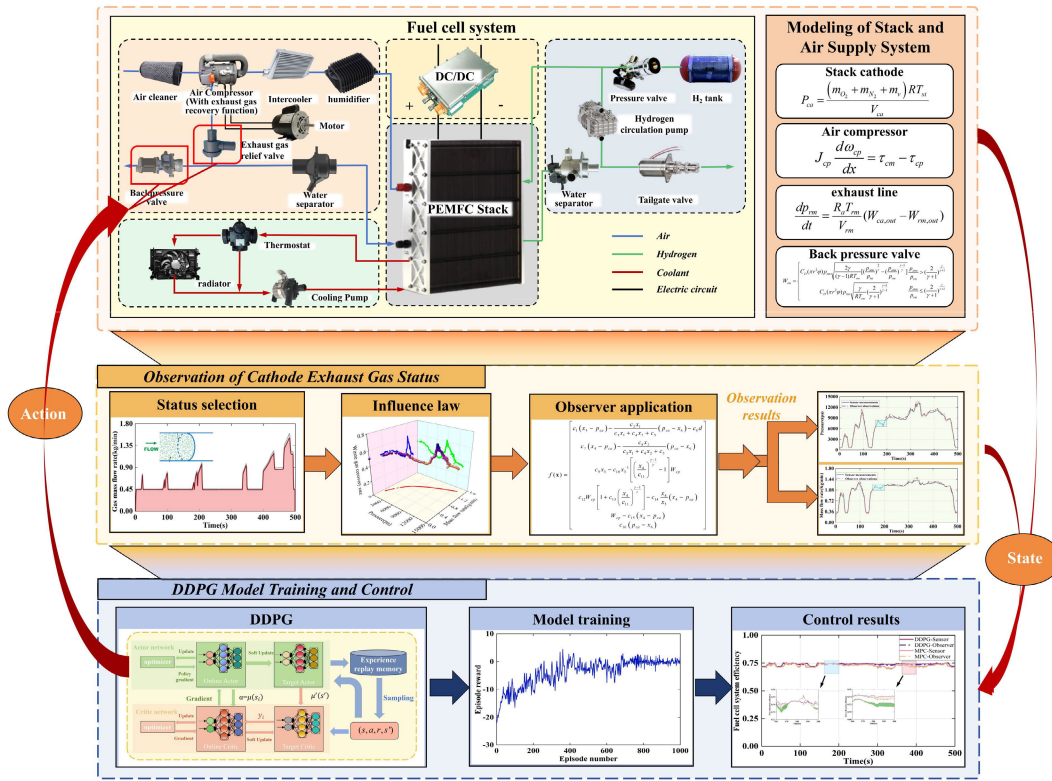


FIGURE 1. Framework of this paper.

voltage, ohmic polarization voltage, activation polarization voltage, and Nernst voltage, the fuel cell's output voltage model expresses the output characteristics of the stack [38]. It is able to be stated as:

$$V_{cell} = V_{Nernst} - V_{ohm} - V_{con} - V_{act} \quad (1)$$

Only the gas pressure and flow at the stack's inlet and outlet are taken into account in the cathode channel model, with changes in channel pressure and flow being disregarded. The main function of the cathode flow field model processing is to dynamically simulate the inlet flow, temperature, humidity, pressure, and other gas characteristic parameters as well as the flow of oxygen, nitrogen, and water vapor into the battery at the cathode side. It is considered that all batteries behave similarly because the battery stack is well designed. One stack cathode volume contains the combined cathode volumes of several fuel cells. The oxygen, nitrogen, and water vapor at the cathode are represented by [39] and [40]:

$$\begin{cases} \frac{dm_{O_2,ca}}{dt} = W_{O_2,in,ca} - W_{O_2,out,ca} - W_{O_2,react} \\ \frac{dm_{N_2,ca}}{dt} = W_{N_2,in,ca} - W_{O_2,out,ca} \\ \frac{dm_{v,ca}}{dt} = W_{v,in,ca} - W_{v,out,ca} + W_{v,gen,ca} + W_{v,mem} \end{cases} \quad (2)$$

The mass flow rate of oxygen, nitrogen, and water reaching the stack's cathode is [41]:

$$\begin{cases} W_{O_2,in,ca} = X_{O_2} \times W_{air,in,ca} = X_{O_2} \times \frac{W_{in,ca}}{1 - \frac{P_{v,in,ca}}{P_{air,in,ca}} \times \frac{M_{H_2O}}{M_{air}}} \\ W_{N_2,in,ca} = (1 - X_{O_2}) \times W_{air,in,ca} = (1 - X_{O_2}) \times \frac{W_{in,ca}}{1 - \frac{P_{v,in,ca}}{P_{air,in,ca}} \times \frac{M_{H_2O}}{M_{air}}} \\ W_{v,in,ca} = W_{in,ca} - W_{O_2,in,ca} - W_{N_2,in,ca} \end{cases} \quad (3)$$

where the mass flow of dry air entering the cathode is denoted by  $W_{air,in,ca}$ ,  $P_{air,in,ca}$  is the air pressure entering the cathode, The oxygen mass percentage in dry air is represented by  $X_{O_2}$ , and the partial pressure of water vapor entering the cathode is denoted by  $P_{v,in,ca}$ .

The mass flow rate of dry air and water vapor flowing out of the cathode is:

$$\begin{cases} W_{air,out,ca} = \frac{1}{1 + \frac{m_{v,ca}}{m_{air,ca}}} W_{out,ca} \\ = \frac{1}{1 + \frac{P_{v,ca}}{P_{air,ca}} \times \frac{M_{H_2O}}{M_{air}}} W_{out,ca} \\ W_{v,out,ca} = W_{out,ca} - W_{O_2,out,ca} - W_{N_2,out,ca} \end{cases} \quad (4)$$

The mass flow rate of the outlet gas can be roughly calculated using the linear nozzle equation due to the cathode cavity's and the cathode outlet pipe's minimal pressure difference [38]:

$$W_{out,ca} = K_{out,ca} (P_{ca} - P_{out,ca}) \quad (5)$$

where  $K_{out,ca}$  are the constant of cathode nozzle and  $P_{out,ca}$  are the pressure of exhaust pipeline.

### B. EGAC MODEL

The EGAC is divided into three parts: compressor assembly, exhaust gas recovery assembly and drive motor. Air enters the compressor from the surrounding environment, and the gas is compressed and delivered to the supply pipeline. The driving motor is responsible for providing necessary power for the air compressor. The gas at the cathode outlet drives the exhaust gas recovery component turbine to rotate through the exhaust gas recovery valve, consequently causing the air compressor to spin quickly. reducing the power demand of the air compressor and saving motor energy.

The real time speed of the air compressor is determined by the driving torque and load torque of the air compressor. Speed characteristic equation of air compressor [39], [42]:

$$\begin{cases} J_{cp} \frac{d\omega_{cp}}{dx} = \tau_{cm} - \tau_{cp} \\ \tau_{cm} = \eta_{cm} \frac{k_t}{R_{cm}} (v_{cm} - k_v \omega_{cp}) \\ \tau_{cp} = \frac{P_{cp}}{\omega_{cp}} \end{cases} \quad (6)$$

where  $J_{cp}$  is the linkage inertia between air compressor and motor.

The power consumption of the drive motor is:

$$P_{cp} = \frac{\gamma}{\gamma - 1} \frac{RT_{atm} W_{cp}}{\eta_{cp}} \left( \left( \frac{P_{cp}}{P_{atm}} \right)^{\frac{\gamma-1}{\gamma}} - 1 \right) \quad (7)$$

where  $p_{cp}$  is the pressure at which the air compressor compresses the gas,  $\eta_{cp}$  is the isentropic efficiency of the air compressor.

The power calculation formula of exhaust gas recovery component is [43] and [44]:

$$P_t = c_{p,ca,out} \eta_{cp} T_{ca,out} \left( \left( \frac{P_{om,out}}{P_{ca,out}} \right)^{1-\frac{1}{\gamma}} - 1 \right) \dot{m}_{ca,out} \quad (8)$$

where  $c_{p,ca,out}$  are the specific heat capacity of the gas at the cathode outlet.

Combined with the cathode channel model and equation (8), the exhaust gas recovery module model can be defined as follows:

$$\begin{aligned} P_e &= c_{p,ca,out} \eta_t T_{ca,out} \frac{I \cdot N_c}{4F} \left( \pi^{1-\left(\frac{1}{\gamma}\right)} - 1 \right) \\ &\times \left( M_{O_2}(S-1) + M_{N_2} S \frac{Y_{N_2}}{Y_{O_2}} \right) \end{aligned} \quad (9)$$

### C. SUPPLY PIPELINE MODEL

The supply pipeline model describes the connecting pipeline between the air compressor and fuel cell stack, including the connecting pipeline between the intercooler and humidifier. The mass flow at the supply pipeline's outlet is a result of the pressure differential between the cathode flow field and the pipeline outlet [38]:

$$W_{sm,out} = k_{sm,out} (p_{sm} - p_{ca}) \quad (10)$$

where  $k_{sm,out}$  is flow coefficient at the supply pipeline's exit is called out. Before the gas enters the cathode flow field from the air compressor, the temperature in the supply pipeline will change. Determine the gas pressure inside the inlet pipe  $p_{sm}$  using the ideal gas state equation [38]:

$$\frac{dp_{sm}}{dt} = \frac{R_a}{V_{sm}} (W_{cp} T_{cp,out} - W_{sm,out} T_{sm}) \quad (11)$$

where  $T_{sm}$  is the temperature of the supply pipeline and  $R_a$  is the ideal gas constant.

### D. RETURN PIPELINE AND BACK PRESSURE VALVE MODEL

The return pipeline model is used to simulate the pipe change between the discharge of mixed gas from the cathode flow field and the back pressure valve. Similar to the gas supply pipeline model, the dynamic equation of the return pipeline can be expressed as [38]:

$$\frac{dp_{om}}{dt} = \frac{RT_{om}}{V_{om}} (W_{ca,out} - W_{om}) \quad (12)$$

where  $p_{om}$  is the pressure in the flue,  $T_{om}$  is the return pipeline temperature, and  $W_{om}$  is the air mass flow rate through the return pipeline.

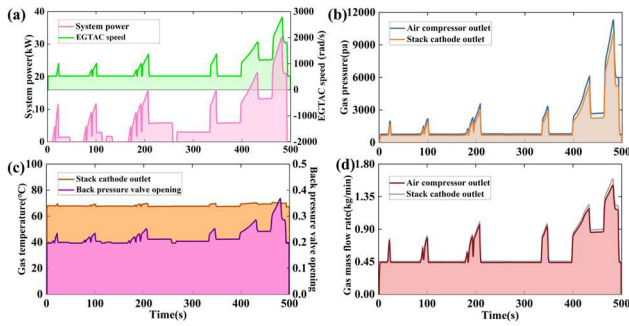
The gas mass flow rate in the return pipeline is:

$$W_{om} = \begin{cases} C_{DA} A_T P_{om} \sqrt{\frac{2\gamma}{(\gamma-1)RT_{om}} \left[ \left( \frac{P_{atm}}{P_{om}} \right)^{\frac{2}{\gamma}} - \left( \frac{P_{atm}}{P_{om}} \right)^{\frac{\gamma-1}{\gamma}} \right]} \\ \frac{P_{atm}}{P_{om}} > \left( \frac{2}{\gamma+1} \right)^{\frac{\gamma}{\gamma-1}} \\ C_{DA} A_T P_{om} \sqrt{\frac{\gamma}{RT_{om}} \left( \frac{2}{\gamma+1} \right)^{\frac{\gamma+1}{\gamma-1}}} \\ \frac{P_{atm}}{P_{om}} \leq \left( \frac{2}{\gamma+1} \right)^{\frac{\gamma}{\gamma-1}} \end{cases} \quad (13)$$

where  $A_T$  is the opening area of the valve,  $T_{om}$  is the gas temperature in the return pipeline, and  $P_{om}$  is the gas pressure in the return pipeline.

In this paper, the back pressure valve's opening  $\theta_{om}$  is controlled to maintain the cathode pressure. Define the back pressure valve opening  $\theta_{om}$  is:

$$\theta_{om} = \frac{A_T}{\pi r_v^2} \times 100, \theta_{om} \in (0, 100) \quad (14)$$



**FIGURE 2.** Simulation results of cathode exhaust and EGTC in fuel cell system (a) system load power and EGTC speed; (b) Outlet gas pressure; (c) Outlet gas temperature and back pressure valve opening; (d) Outlet gas pressure mass flow rate.

**E. FUEL CELL SYSTEM POWER AND EFFICIENCY**

A fuel cell stack consists of multiple individual cells that are interconnected in a series configuration. The stack power is:

$$P_{st} = n_{st} V_{cell} I_{st} \tag{15}$$

A vehicle fuel cell system comprises an air supply system, a hydrogen supply system, a thermal management system, and an electric stack. The EGTC is the component that consumes the most power in the fuel cell system. Parasitic elements such as the hydrogen circulation pump, water pump, and cooling fan also contribute to power consumption. However, when compared to the power consumption of the EGTC, their impact on the output power of the fuel cell system is minimal and can be disregarded. Hence, this study just focuses on the power consumption of the EGTC, disregarding the impact of other parasitic components. The system output power is determined by subtracting the power consumption of parasitic components from the effective output power of the stack [45]:

$$P = P_{st} - P_{cp} \tag{16}$$

where  $P_{CP}$  is the power of parasitic components. The electric efficiency of fuel cell system is [46]:

$$\eta_{fc} = \frac{P}{P_{st}} \tag{17}$$

The main parameters in the model of the of PEMFC satek model and air supply system are shown in Table 1.

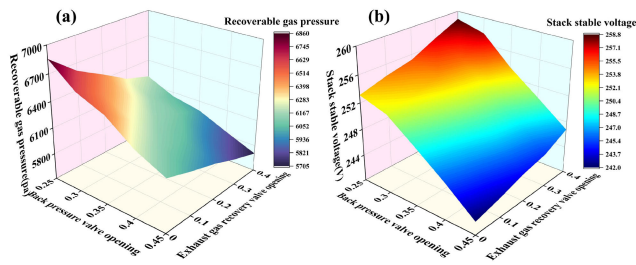
**F. MODEL SIMULATION RESULTS**

Based on the aforementioned mathematical model, this study employs Amesim software to construct a simulation model of a fuel cell system that includes EGTC. The system undergoes 500s variable load power simulation, and simulation results are displayed in Figure 2. Figure 2(a)'s pink curve represents the applied load power of fuel cell system, while the simulation outcome of the EGTC speed is depicted by the green curve. The alteration of the EGTC's velocity is essentially in accord with the modification of load power in the fuel cell system. The reason is that an elevation in the load

**TABLE 1.** Main parameters of PEMFC satek model and air supply system model.

Parameter	Unit	Valve
Stack operating temperature $T_{st}$	K	352
Number of single cells $n_{st}$		382
Faraday constant $F$		96485
Reaction area $A_{act}$	$m^2$	0.02
Designed power of stack $P_{fc}$	kW	115
Volume of cathode flow field $V_{ca}$	$m^3$	0.01
Nozzle constant of cathode $K_{ca,out}$	$kg/(s \cdot Pa)$	$2.1776 \times 10^{-6}$
Molar mass of oxygen $M_{O_2}$	kg/mol	0.032
Molar mass of nitrogen $M_{N_2}$	kg/mol	0.028
Molar mass of water $M_{H_2O}$	kg/mol	0.018
Molar mass of dry air $M_{air}$	kg/mol	0.029
Gas constant of air $R_a$	J/(kg·K)	298.6
adiabatic exponent of air $\gamma$		1.4
Specific Heat Capacity of Air $C_p$	J/kg·K	1004.88
Motor torque sensitivity constant $k_t$	N·m/A	0.0153
Motor back EMF constant $k_v$	V/(rad/sec)	0.0153
Armature resistance $R_{em}$	$\Omega$	0.9
Air compressor inlet air temperature $T_{atm}$	K	298.15
Air compressor inlet air pressure $P_{atm}$	Pa	101,325
Flow coefficient of the supply manifold outlet $k_{sm,out}$		$3.629 \times 10^{-6}$
Volume of the supply manifold $V_{sm}$	$m^3$	0.02
The flow coefficient of the reflux pipeline $C_D$		0.0124
Volume of the return pipeline $V_{om}$	$m^3$	$1.52 \times 10^{-3}$
Back pressure valve radius $r_v$	m	0.0228

power necessitates an increase in gas supply for the system, and correspondingly, the EGTC must generate a higher gas flow to meet this requirement. As a result, the velocity is adjusted accordingly. The findings of the simulation for the gas pressure at the EGTC's outlet and the stack's cathode are exhibited in Figure 2(b). The gas pressure at both the EGTC's outlet and the stack's cathode is subject to change in line with variations in load power. This is done to satisfy the cathode's requirement for an adequate supply of oxygen. The simulation results for the opening of back pressure valve and the gas temperature at the cathode outlet of the stack are presented in Figure 2(c). By controlling the gas flow's resistance at both the cathode outlet and exhaust pipe, the valve's opening will increase in proportion to the load power rise. The back pressure valve's opening can stabilize the gas flow and pressure, yet the temperature at the cathode outlet of the stack remains stable at approximately 67°C; Figure 2(d) displays the simulation findings for the gas mass flow rate at the cathode outlet of the EGTC and stack. The gas mass flow rate at the outlet of the EGTC will change correspondingly with the load power variation as the outlet gas pressure changes. Therefore, the cathode outlet pressure and fuel cell mass flow rate have been chosen as parameters for estimating



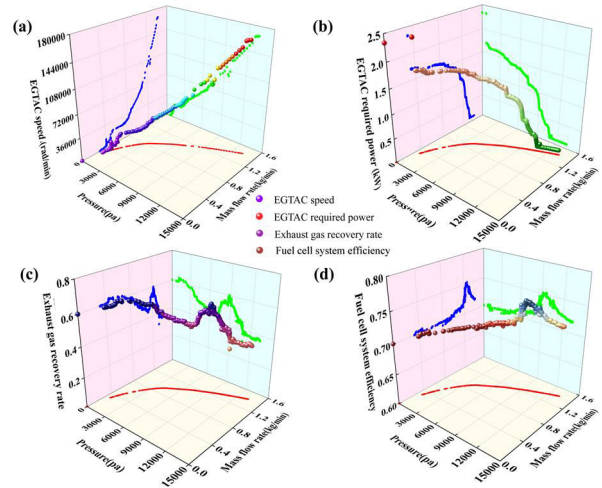
**FIGURE 3.** Effect of valve opening on (a) recoverable gas pressure; (b) Stable voltage of stack.

research and exhaust characteristics which enables reflection and analysis of the performance and working efficiency of the EGAC and fuel cell system under diverse load power.

### III. CONTROL PARAMETER IMPACT ANALYSIS AND ESTIMATION OF CATHODE EXHAUST CHARACTERISTICS

#### A. EFFECT OF VALVE OPENING ON EXHAUST ENERGY AND FUEL CELL PERFORMANCE

To study the influence of the back pressure valve and exhaust gas recovery valve opening on the operation of the stack and the EGAC, this paper selects the working condition of the fuel cell system loaded at 60% of the rated power for the study, and sets the back pressure valve opening to 0.25, 0.3, 0.35, 0.4 and 0.45 in order, and the exhaust gas recovery valve opening to 0, 0.1, 0.2, 0.3, 0.4, the recoverable exhaust gas of the EGAC and the stable voltage of the stack under different opening of the two valves are shown in Figure 3: Figure 3.(a) is the EGAC can be recovered gas pressure with the change of these two valve openings, when the exhaust gas recovery valve opening is kept constant at 0, with the backpressure valve opening is reduced from 0.45 to 0.25, the EGAC can be recovered gas pressure increases from 6102.3Pa to 6856.7Pa. When the 0.45 backpressure valve opening remains constant, the recoverable gas pressure of the EGAC decreases from 6102.3 Pa to 5704.5 Pa as the exhaust gas recovery valve opening increases from 0 to 0.4; Figure3(b) shows the variation of the fuel cell stack's stabilized voltage with the change of these two valve openings When the backpressure valve opening is kept constant at 0.45, the stable voltage of the stack increases from 242.1V to 246.8V as the exhaust gas recirculation valve opening increases from 0 to 0.4. When the backpressure valve opening is kept constant at 0.25, the stable voltage of the stack increases from 242.1V to 253.2V as the exhaust gas recirculation valve opening increases from 0 to 0.4. This is because when the back pressure valve's opening is reduced, the resistance of exhaust gas emission is increased, and the gas pressure in the stack's cathode outlet, i.e. the exhaust pipeline, is increased which allows the EGAC to recover the energy in the flue gas more effectively, increasing the amount of air entering the stack and improving stack performance. Similarly, if the exhaust gas recirculation valve opening is increased, the EGAC will start to recirculate the exhaust gas, and the gas



**FIGURE 4.** Effect of exhaust gas energy recovered by the EGAC on (a) EGAC speed; (b) Required power of EGAC; (c) Exhaust gas recovery rate; (d) fuel cell system efficiency.

at the cathode outlet of the stack will decrease, and then the air entering the stack will increase, and the performance of the stack will also improve. However, if the opening of the backpressure valve is set too small for a long time, the cathode reaction gas will not be discharged, which can easily damage the internal structure of the stack.

#### B. EFFECT OF RECOVERED EXHAUST GAS ENERGY ON FUEL CELL SYSTEM PERFORMANCE

To further investigate the specific effect of exhaust gas energy recovered by the EGAC on the performance of the EGAC and fuel cell system, this paper chooses to increase the gas pressure recovered by the EGAC from 655.52 Pa to 13233.89 Pa, and the mass flow rate of recovered gas from 0.32356 kg/min to 1.4179 kg/min. The performance of the EGAC and the stack under the change of exhaust gas energy recovered by the EGAC is shown in Figure 4: Figure 4(a) shows that with the increase of extracted energy from the exhaust gas by the EGAC, the EGAC speed is almost linear with increasing speed; Figure 4(b) demonstrates that when the energy of the exhaust gas that the EGAC recovers rises, the power required by the EGAC gradually decreases; at the beginning of exhaust gas recovery by the EGAC, that is, when the gas pressure is less than 8000 Pa, the EGAC still needs to be driven by the engine, and the power required by the EGAC decreases relatively slowly, when the energy of the exhaust gas increases, that is, when the gas pressure reaches 9000 Pa, the power required by the EGAC decreases relatively rapidly, but when the energy of the exhaust gas is sufficiently high, that is, when the gas pressure is greater than 12,000 Pa, the power required by the EGAC does not continue to decrease; Define the calculation formula of the energy recovery efficiency of the exhaust gas of the EGAC as:  $\eta_e = W_1/W_0$ . Where  $W_1$  is the mass flow at the outlet of the exhaust gas recovery comp,

$W_0$  is the mass flow at the EGTA exhaust gas recovery component's input. Figure 4(c) shows that the efficiency of EGTA gas recovery gradually decreases as the energy of EGTA gas recovery increases. When the exhaust pressure reaches 7769.34 Pa, the efficiency of the EGTA exhaust gas recovery increases and continues to decrease after reaching the local peak when the exhaust pressure reaches 8949.98 Pa; Figure 4(d) shows that the efficiency of the fuel cell system slowly increases from 70.1% with the increase of the energy recovered from the exhaust gas by the EGTA. When the exhaust pressure reaches 7713.67 Pa, the efficiency of the fuel cell system increases rapidly reaching about 75.3%, and starts to decrease after reaching the maximum value of the fuel cell system when the exhaust pressure reaches 9127.46 Pa. This is because as the recovered exhaust energy increases, it can be used to provide more power, drive the EGTA to rotate faster, reduce the power required by the EGTA, and thus improve the fuel cell system's efficiency. However, the capacity of the EGTA to recover exhaust gas is limited, so the efficiency of exhaust gas recovery is reduced. When the energy recovered from the exhaust gas is sufficient, the exhaust gas recovery system is saturated and the power consumption of the EGTA has reached a stable state, at which time more exhaust gas energy cannot be effectively used, and the efficiency of the fuel cell has decreased, but it is still higher than the efficiency of the system without exhaust gas recovery.

### C. OBSERVATION OF GAS STATE AT CATHODE OUTLET OF FUEL CELL

Owing to the fuel cell's broad working range, variations in mass flow and gas pressure at the stack's cathode outlet will result in variations in the exhaust gas recirculation system's output, and it is easy to render the exhaust gas recirculation system ineffective. Based on the above, the exhaust gas supply directly affects the output of the EGTA. If the oxygen supply is insufficient, the energy generated by the exhaust gas cannot meet the power demand of the EGTA; if the exhaust gas supply is too high, the parasitic power of the exhaust gas recovery component increases, affecting the efficiency of the exhaust gas recovery. Exhaust energy control refers to the control of gas pressure and mass flow in the channel at the cathode outlet side of the stack. However, it is challenging to measure the pressure at the cathode outlet of the stack  $p_{ca,out}$ . Therefore, the soft measurement method is adopted, i.e. Realtime pressure estimates are intended to be provided by the state observer, and the mass flow is estimated according to the cathode pressure estimated in real time.

Based on the model established in section II, the model of the fuel cell air supply system can be written as a nonlinear state space equation, as shown in the formula:

$$\begin{cases} \dot{x} = g(x) \cdot u + f(x) \\ y = Cx \end{cases} \quad (18)$$

where, state  $x \in R^6$ ,  $x = [m_{o2,ca} \ m_{N2,ca} \ \omega_{cp} \ W_{sm,out} \ p_{sm} \ p_{om}]^T$ , model input  $u \in R^2$ ,  $u = [\theta_e \ \theta_{rm}]^T$ , model input

$y \in R^2$ ,  $y = [\omega_{cp} \ p_{sm}]^T$ , can be measured by sensor.  $C$  is the measurement matrix. Matrices  $f(x)$  and  $g(x)$  can be written as (19) and (20), as shown at the bottom of the next page.

Regarding the state  $p_{ca,out}$  to be observed as a new state  $x_7$ , the equation of state  $x_6$  can be rewritten as:

$$\begin{cases} \dot{x}_6 = k_{sm} \frac{V_{om}}{R_a} x_7 - k_{sm} \frac{V_{cm}}{R_a} x_6 + g(x_6) \cdot u_2 \\ \dot{x}_7 = \mu(t) \end{cases} \quad (21)$$

where,  $g(x_6) = k_{ca,out} R_a T_{om} W_{om} x_6 / V_{om}$ . Due to the complexity of the dynamic characteristics of  $p_{ca,out}$ , its variation is temporarily represented by  $\mu(t)$ , and the observer is designed as:

$$\begin{cases} \dot{\bar{x}}_6 = k_{sm} \frac{V_{om}}{R_a} \bar{x}_7 - k_{sm} \frac{V_{cm}}{R_a} \bar{x}_6 + \alpha_1 (x_6 - \bar{x}_6) \\ \dot{\bar{x}}_7 = \alpha_2 (x_6 - \bar{x}_6) \end{cases} \quad (22)$$

where,  $\bar{x}_6$  and  $\bar{x}_7$  are the estimated values of states  $x_6$  and  $x_7$  respectively,  $\alpha_1$  and  $\alpha_2$  is the parameter to be determined. Through Simulink simulation, the values of  $\alpha_1$  and  $\alpha_2$  are determined to be 50 and 450. Therefore, we can get the estimated values of the observed  $p_{ca,out}$  and  $W_{ca,out}$ :

$$\begin{cases} \hat{p}_{ca,out} = \bar{x}_7 \\ \hat{W}_{ca,out} = \chi_{v,in} k_{sm} (x_6 - \hat{p}_{ca,out}) / \frac{nM_v I_{st}}{4F} \end{cases} \quad (23)$$

To verify the accuracy of the state observer, simulation tests were performed on the Simulink platform. Figures 5 to 7 show the comparison results of the estimation of the cathode outlet pressure and mass flow of the fuel cell system in various load scenarios. The estimation results of the gas characteristics at the cathode outlet of the stack under high load power are shown in Figure 5: Figure 5(a) shows the applied load power, and Figure 5(b) and Figure 5(c) show the observation and comparison results of the cathode outlet pressure and mass flow rate of the fuel cell under this load, which can accurately estimate the exhaust gas characteristics. From the local enlarged view, it is more obvious that the designed observer can better follow the measured values of the sensor, Figure 5(d) and Figure 5(e) show the estimation error of pressure and mass flow rate under this load. The maximum error rate of pressure estimation is less than 13.84% and the maximum error rate of mass flow estimation is less than 12.10%.

The estimation results of the gas characteristics at the cathode outlet of the stack under medium load power are shown in Figure 6: Figure 6(a) shows the applied load power, and Figure 6(b) and Figure 6(c) show the observation and comparison results of the cathode outlet pressure and mass flow rate of the fuel cell under this load. It is more obvious that the designed observer can better follow the measured value of the sensor from the local enlarged view, and Figure 6(d) and Figure 6(e) show the estimation error of pressure and mass flow rate under this load. The maximum error rate of pressure estimation is less than 13.23%, and the maximum error rate of mass flow estimation is less than 11.60%.

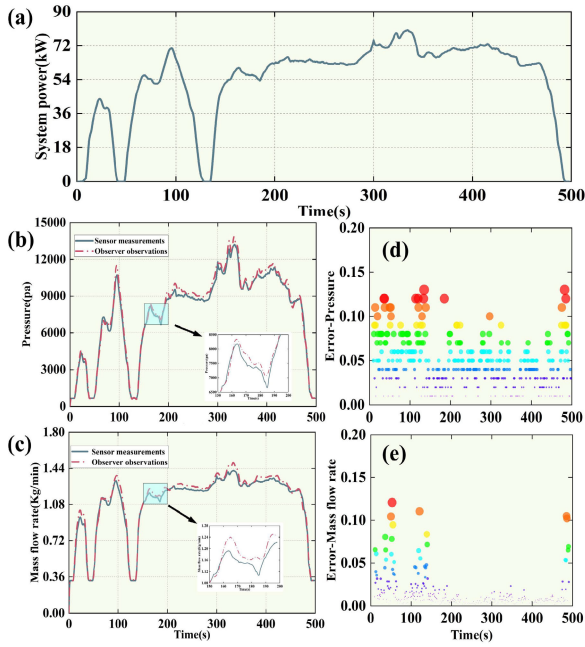


FIGURE 5. Comparison of gas state sensor and observer at cathode outlet of stack under high load power.

The estimation results of the gas characteristics at the cathode outlet of the stack under low load power are shown in Figure 7: Figure 7(a) shows the applied load power, and Figure 7(b) and Figure 7(c) show the observation and comparison results of the cathode outlet pressure and mass flow rate of the fuel cell under this load. It is more obvious that the designed observer can better follow the measured value of the sensor from the local enlarged view, Figure 7(d) and Figure 7(e) show the estimation error of pressure and mass flow rate under this load. The maximum error rate of pressure estimation is less than 11.63%, and the maximum error rate of mass flow rate estimation is less than 10.89%. Therefore, the

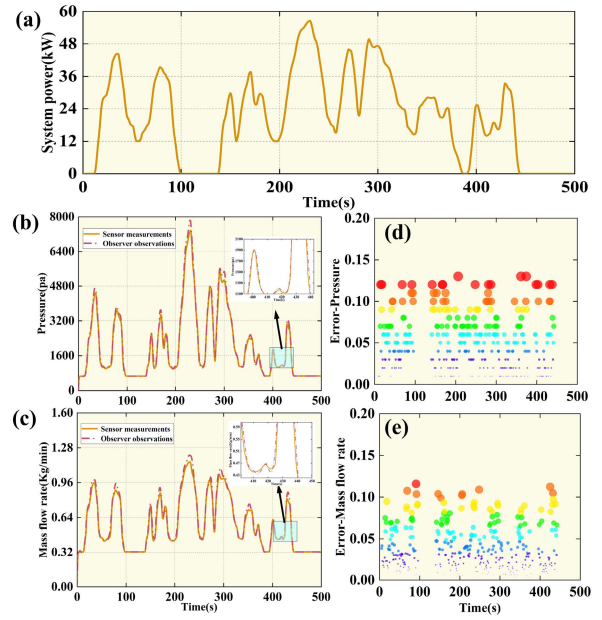


FIGURE 6. Comparison of gas state sensor and observer at cathode outlet of stack under medium load power.

designed observer can track the gas pressure at the cathode outlet well and the estimation error is within the acceptable range.

#### IV. EGATC ENERGY RECOVERY OPTIMIZATION INTELLIGENT CONTROL METHOD

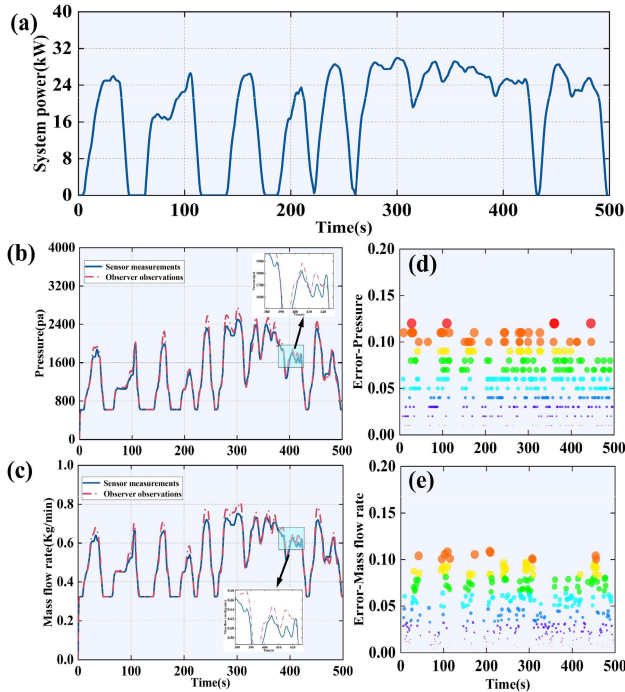
##### A. CONTROL STRUCTURE OF THE EGATC BASED ON DDPG ALGORITHM

This paper proposes an intelligent control architecture of the EGATC founded on the deep deterministic policy gradient (DDPG), as shown in Figure 8(a). The framework takes the PEM fuel cell system as the environment and the DDPG algorithm as the intelligent agent. The goal is to improve the

$$f(x) = \begin{bmatrix} \frac{k_{ca}M_{O_2}x_1}{M_{O_2}x_1+M_{N_2}x_2+M_v} (x_6 - p_{ca,out}) + k_{sm}\chi_{O_2} (x_5 - p_{ca,out}) \\ \frac{k_{ca}M_{N_2}x_2}{M_{O_2}x_1+M_{N_2}x_2+M_v} (x_6 - p_{ca,out}) + k_{sm}\chi_{N_2} (x_5 - p_{ca,out}) \\ \frac{\eta_{cm}k_l k_v}{J_{cp}R_{cm}} x_3 - \frac{C_p T_{atm}}{J_{cp}\eta_{cp}} x_3^{-1} \left[ \left( \frac{x_5}{p_{atm}} \right)^{\frac{\gamma-1}{r}} - 1 \right] W_{cp,out} \\ k_{sm} \frac{x_5}{x_4} (p_{ca,out} - x_5) + \frac{\gamma R_a T_{atm} W_{cp,out}}{V_{sm}} \left[ 1 + \frac{1}{\eta_{cp}} \left( \frac{x_5}{p_{atm}} \right)^{\frac{\gamma-1}{r}} \right] \\ W_{cp,out} - k_{sm} (x_5 - p_{ca,out}) \\ k_{sm} \frac{V_{cm}}{R_a} (p_{ca,out} - x_6) \end{bmatrix} \quad (19)$$

$$g(x) = \begin{bmatrix} 0 & 0 \\ 0 & 0 \\ \frac{\eta_{cm}k_l}{J_{cp}R_{cm}} & 0 \\ 0 & 0 \\ 0 & 0 \\ 0 & -k_{ca} \frac{R_a T_{om}}{V_{rm}} W_{om,out} \end{bmatrix} \quad (20)$$





**FIGURE 7.** Comparison of gas state sensor and observer at cathode outlet of stack under low load power.

fuel cell system’s efficiency while improving the efficiency of the EGATC exhaust gas recovery. The algorithm flow of DDPG is shown in Table 2.

In DDPG algorithm, state and action are important components of learning.

The state quantity  $s_t$  is expressed as:

$$s_t = \{P_{cp}, W_{rm}, p_{rm}, p_{ca,out}, W_{ca,out}\} \quad (24)$$

The energy of exhaust gas entering the EGATC is managed by modifying the exhaust gas recovery valve’s and back pressure valve’s opening, so the action  $a_t$  is:

$$a_t = \{\theta_{BPV}, \theta_{EGR}\} \quad (25)$$

The reward plays the role of guiding the agent to learn and optimize strategies. The reward  $r$  is set as:

$$R = \{k_1 \eta_e^2 + k_2 \eta_{fc}^2\} \quad (26)$$

where  $k_1$  and  $k_2$  are weight coefficients.

### B. DDPG ALGORITHM TRAINING AND TRAINING RESULTS ANALYSIS

DDPG is an optimization algorithm that based on a deterministic strategy. The algorithm’s training process involves sending the state variables of the fuel cell system (such as the power consumption of the driving motor, the gas pressure through the Return pipeline, the gas mass flow through the Return pipeline, the gas pressure at the cathode outlet, and the gas mass flow at the cathode outlet) to the DDPG training environment to train the agent. The Actor network determines

**TABLE 2.** DDPG algorithm flow.

DDPG algorithm flow	
<p><i>Input:</i> policy current network parameters <math>\theta^p</math>, Policy target network parameters <math>\theta^p</math>, Evaluate current network parameters <math>\theta^p</math>. Evaluation target network parameters <math>\theta^p</math>, attenuation factor <math>\gamma</math>, Soft renewal coefficient <math>\tau</math>, The number of empirical playback samples <math>n</math>, the target network parameter update frequency <math>f</math>, the number of iteration rounds <math>M</math>, the number of single round iterations <math>T</math>, and the random process <math>N</math></p> <p><i>Output:</i> optimal policy current network parameters <math>\theta^p</math> and evaluate current network parameters <math>\theta^p</math></p> <ol style="list-style-type: none"> <li>1. random initialization <math>\theta^p, \theta^p</math>, Concurrent order <math>\theta^p = \theta^p, \theta^p = \theta^p</math>.</li> <li>Initialize experience playback pool <math>P</math></li> <li>2. For rounds = 1 : <math>M</math> do</li> <li>Obtain initial state <math>S_1</math></li> <li>For <math>t = 1 : T</math> do</li> <li>a) Based on the current network of the policy, enter the state <math>S_t</math> to get the action <math>A_t = \mu(S_t) + N_t</math></li> <li>b) Execute action <math>A_t</math> to get new status <math>S_{t+1}</math> and reward <math>R_t</math></li> <li>c) Store <math>(S_t, R_t, A_t, S_{t+1})</math> in <math>P</math></li> <li>d) Let <math>S_t = S_{t+1}</math></li> <li>e) Randomly sample <math>n</math> samples <math>(S_i, R_i, A_i, S_{i+1}), i = 1, 2, \dots, n</math>, and calculate the current target <math>Q</math> value <math>y_i, y_i = R_i + \gamma Q^*(S_{i+1}, \mu^*(S_{i+1}), \theta^p</math></li> <li>f) Based on the mean square loss function,</li> <math display="block">L = \frac{1}{n} \sum_{i=1}^n (y_i - Q(S_i, A_i, \theta^p))^2</math> <li>the parameters of the current network are updated by gradient back propagation <math>\theta^p</math></li> <li>g) Based on the loss function, <math>J(\theta^p) = -\frac{1}{n} \sum_{i=1}^n (Q(S_i, A_i, \theta^p))</math> the parameters of the current network are updated by gradient back propagation strategy <math>\theta^p</math></li> <li>h) When <math>M\%f = 1</math>, update the evaluation target network and strategy target network parameters:</li> <math display="block">\theta^{p*} = \tau \theta^p + (1 - \tau) \theta^{p*}</math> <math display="block">\theta^{p*} = \tau \theta^{p*} + (1 - \tau) \theta^{p*}</math> <li>end for</li> <li>end for</li> </ol>	

the action, (i.e. the opening of the back pressure valve and the exhaust gas recovery valve), based on the current state. It then feeds the current action and state into the Critic network to calculate the  $Q$  value. Finally, the Actor network parameters are updated using gradient ascent. then, the action value is assigned to each component as the provided value, and then, the matching reward and penalty value is acquired through the utilization of the specified reward and penalty function. Simultaneously, the subsequent state is derived depending on the current state. The Critic network updates its parameters using gradient descent between the current action and state, and the future action and state. This process is described by the following formula:

$$L(\theta^Q) = E \left[ \left( r + \gamma Q_T(s', a' | \theta_T^Q) - Q(s, a | \theta^Q) \right)^2 \right] \quad (27)$$

$$\nabla_{\theta^\mu} Q \approx E \left[ \nabla_a Q(s, a | \theta^Q) \Big|_{a=\mu(s)} \cdot \nabla_{\theta^\mu} \mu(s | \theta^\mu) \right] \quad (28)$$

TABLE 3. Units for magnetic properties.

Parameter	Parameter value
Training rounds episodes	1000
Memory pool capacity $N$	10000
Minimum sample set sample	36
Number $n$	
Actor learning rate	0.002
Critic learning rate	0.0012
Discount rate	0.99

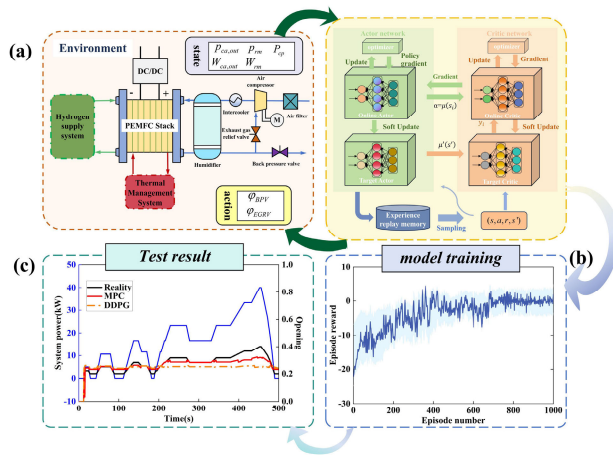


FIGURE 8. EGTAC energy recovery control architecture based on DDPG algorithm.

where  $L(\theta^Q)$  is the average loss function;  $\theta^\mu$  and  $\theta^Q$  is the parameter of strategy network Actor and evaluation network Critic respectively;  $\theta_T^Q$  is the parameter of target Critic;  $\gamma \in [0, 1]$  is the discount factor;  $Q(s, a | \theta^Q)$  is the Q value of the current state-action pair;  $Q_T(s', a' | \theta_T^Q)$  is the Q value of the next time state-action pair;  $\mu(s, a | \theta^\mu)$  is the policy output from the Actor network, where the target network structure is the same as the current network structure. When the average loss function converges, the algorithm is trained. At this time, the Actor network is the optimal strategy, which can realize the real-time control of the energy recovery of the EGATC.

Before the training is officially started, it is necessary to build a connection network and explain the training parameters of the agent agent. The training parameter values of DDPG are shown in Table 3.

Figure 8(b) depicts the round awards and average rewards of the EGATC exhaust gas energy recovery control method training. The algorithm takes 4.5 hours to train 1000 rounds. The single reward tends to converge during 780 training rounds. The round reward and the average reward are gradually improved during continuous training, which shows that the algorithm gradually improves the reward performance in the learning process. The training results show that the parameter setting and network structure of the DDPG algorithm are reasonable.

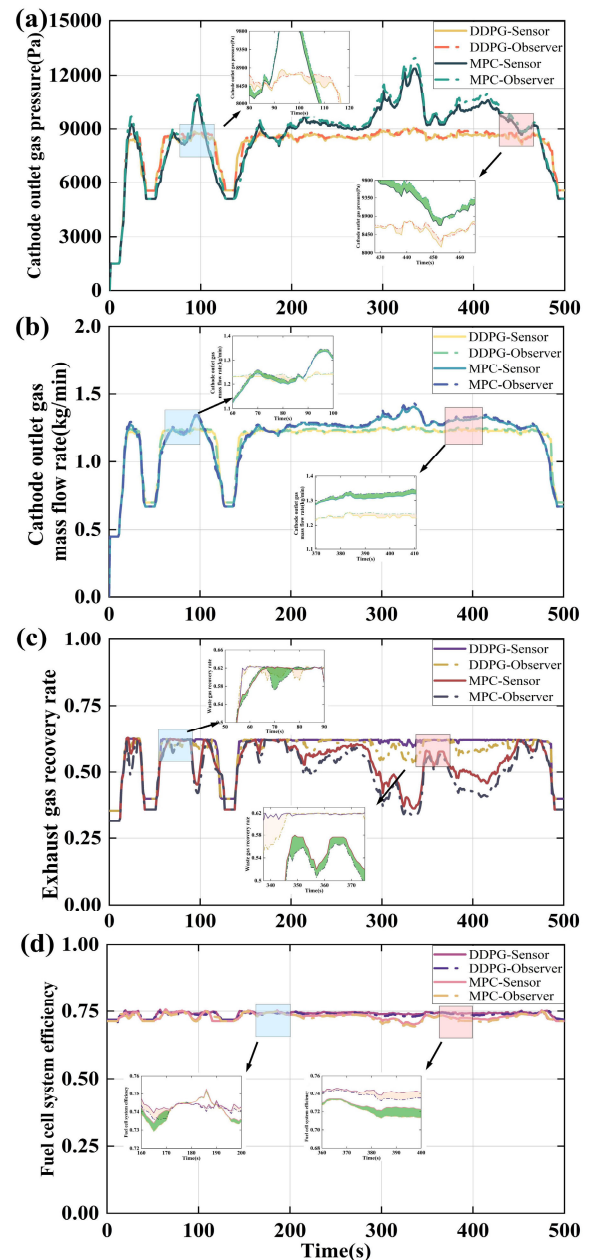


FIGURE 9. Control results under high-power condition (a) cathode outlet gas pressure; (b) Gas mass flow rate at cathode outlet; (c) EGTAC exhaust gas recovery rate; (d) Fuel cell system efficiency.

C. CONTROL METHOD TEST RESULTS

In this work, the variable load data of the fuel cell system is selected as the test condition, as shown in Figure 8(c). Compare the backpressure valve’s opening with the MPC control. When the fuel cell system increases and decreases the load, the MPC cannot control the back pressure valve’s opening well, and instability will occur; however, under the control of the DDPG, the opening of the back pressure valve can be accurately and stably controlled in the target range, which has good robustness.

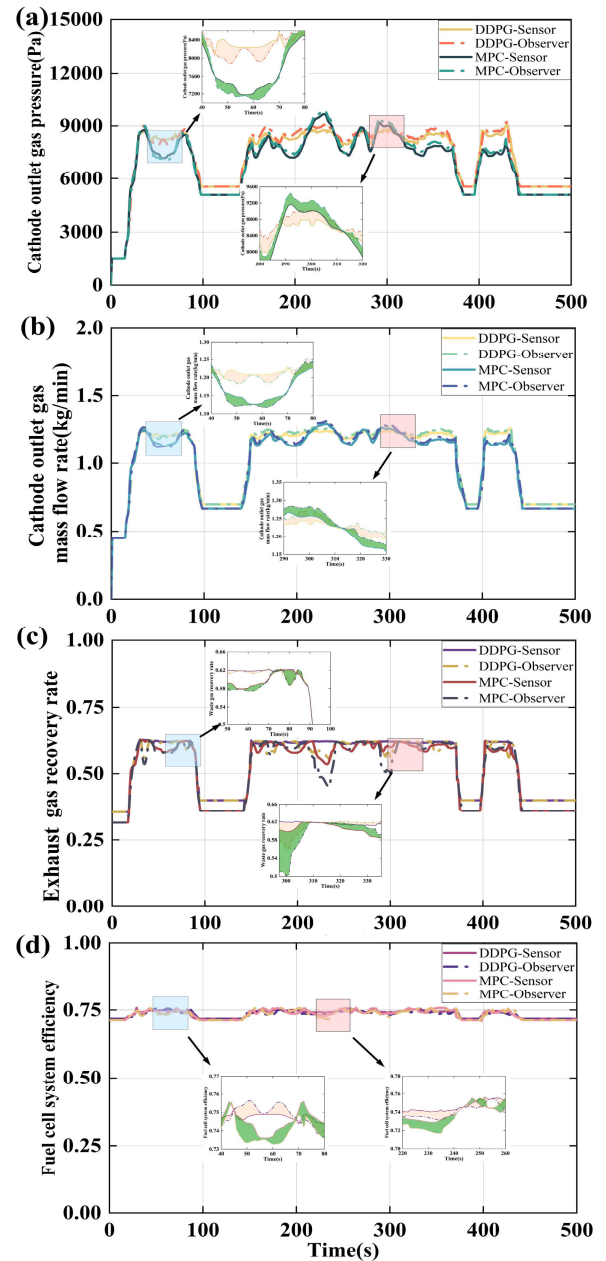
#### D. RESULT ANALYSIS

The control outcomes of two different control algorithms based on sensor and observer under high power load conditions for exhaust gas pressure and gas mass flow rate entering the EGTC are displayed in Figures 9(a) and 9(b). With the modification of load power, the gas pressure governed by the sensor and observer under the DDPG essentially maintains stability around 8540.43 Pa, the gas pressure observed by the observer is higher than that measured by the sensor in most control times within 500s, and the maximum error between the two is less than 296.53 Pa. The gas mass rate remains essentially stable at around 1.22 kg/min, the gas mass flow rate observed by the observer is also higher than that measured by the sensor in most control times within 500s, and the maximum error between the two methods is less than 0.09 kg/min. Nevertheless, when subjected to the MPC, the exhaust gas pressure and mass flow rate exhibit significant fluctuations. However, when comparing the results obtained from the control based on sensor and observer under the MPC, the two methods' error is also small.

The results of controlling and optimizing the EGTC exhaust gas recovery rate and the fuel cell system efficiency are presented in Figures 9(c) and 9(d) respectively. As the load power changes, the sensor-based EGTC exhaust gas recovery rate remains stable at over 60.1% under the DDPG, which is 21.5% higher than that of the MPC. However, the optimization effect of the observer-based EGTC exhaust gas recovery rate is worse than that of the sensor-based EGTC when controlled by the DDPG, the maximum difference is less than 6.5%. Furthermore, the efficiency of the sensor-based fuel cell system when controlled by the DDPG remains stable at over 74.2%, which is 3.6% higher than that seen with the MPC. The optimization efficiency of fuel cell system based on observer controlled by the DDPG is also inferior to that of the optimization efficiency fuel cell system based on sensor. Nevertheless, the maximum difference is less than 2.0%.

Under the condition of high power load, the gas pressure and gas mass flow rate observed by the observer are higher than those measured by the sensor under the same algorithm control. The difference arises from the observer's mistake in monitoring the exhaust gas characteristics. Consequently, the optimization of the EGTC exhaust gas recovery rate and fuel cell system efficiency, when controlled by the DDPG based on the observer, is inferior to that based on the sensor. However, the difference is small, therefore indicating that the proposed sensorless control method is really efficient in the condition of high power load.

The control outcomes of two different control algorithms based on sensor and observer under medium power load conditions for exhaust gas pressure and gas mass flow rate entering the EGTC are displayed in Figures 10(a) and 10(b). With the modification of load power, the gas pressure governed by the sensor and observer under the DDPG essentially maintains stability around 8528.81 Pa, different from the case under high power load condition, the gas



**FIGURE 10.** Control results under medium power condition (a) cathode outlet gas pressure; (b) Gas mass flow rate at cathode outlet; (c) EGTC exhaust gas recovery rate; (d) Fuel cell system efficiency.

pressure observed by the observer and that measured by the sensor are high and low within 500s, but the maximum error between the two is less than 322.90 Pa. The gas mass rate remains essentially stable at around 1.22 kg/min, the gas mass flow rate observed by the observer and that measured by the sensor are also high and low within 500s, but the maximum error between the two is less than 0.15 kg/min. Similar to the high power load condition, the fluctuation of exhaust gas pressure and mass flow rate under the control of MPC is higher than that of DDPG. However, compared with the control results obtained using sensors and observers, the two methods' error is also low. The

results of controlling and optimizing the EGTAC exhaust gas recovery rate and the fuel cell system efficiency are presented in Figures 10(c) and 10(d) respectively. As the load power changes, the sensor-based EGTAC exhaust gas recovery rate remains stable at over 61.08% under the DDPG, which is 13.5% higher than that of the MPC, the optimization of the observer-based EGTAC exhaust gas recovery rate, under the DDPG, is generally inferior to that of sensor-based optimization. However, there are certain periods, particularly when there is a sudden change in gas pressure change rate, where the observer-based optimization outperforms the sensor-based optimization. The maximum difference is less than 5.8%. Furthermore, the efficiency of the sensor-based fuel cell system under the DDPG remains stable at over 74.3%, which is 2.8% higher than that seen with the MPC. Similar to the results of exhaust gas recovery rate, the efficiency optimization effect of observer-based fuel cell system under the DDPG is worse than that of sensor-based optimization in most periods, but better than that of sensor based optimization in some periods, and the maximum difference is less than 1.4%.

Under the medium power load condition, the gas pressure and gas mass flow rate observed by the observer may either exceed or fall below the values measured by the sensor, under the same algorithm. The reason for this result is that the applied medium power load changes more frequently, and the estimation of cathode gas characteristics of fuel cell system has deviation. Nevertheless, the fuel cell's cathode exhaust is reduced during medium power load conditions, resulting in a smaller error in estimating gas characteristics compared to high power load conditions. Consequently, the optimization of the EGTAC exhaust gas recovery rate and fuel cell system efficiency based on observer under the same algorithm control is sometimes better than that based on sensor, and sometimes worse than that based on sensor, but the difference is smaller than that under high power load condition. Hence, the proposed sensorless control method is efficient in the condition of medium power load. Because the exhaust gas energy of the system is less under the low power load condition, this paper does not consider its control. Furthermore, under high power load conditions and medium power load conditions, compared with the electrical efficiency results of the fuel cell system obtained by controlling the air supply system to optimize the response characteristics and power density of the fuel cell in literature [47], [48], [49], the efficiency of the fuel cell system optimized based on the cathode exhaust gas energy recovery control in this study is at a reasonable level. According to the study of the control optimization findings presented above, the sensorless estimating method proposed can substitute the sensor function and significantly enhance the efficiency of exhaust energy recovery and fuel cell system.

## V. CONCLUSION

This study examines how the recovery of energy from the EGTAC affects fuel cell system performance. Additionally,

a method for estimating the exhaust gas characteristics at the cathode outlet of the stack is proposed, and a sensorless control method for exhaust gas energy recovery, utilizing reinforcement learning algorithms, is designed to optimize the performance of both the EGTAC and the fuel cell system. From these workings, conclusions are drawn.

(1) When the back pressure valve's opening decreases during loading of the PEMFC, it causes an increase in gas flow and pressure entering the EGTAC due to the rise in cathode exhaust gas from the stack which reduces the power requirement of the EGTAC. With an increase in exhaust pressure from 7713.67 Pa to 9127.46 Pa, the effectiveness of fuel cell system improves by 3.2%. However, the energy required by the EGTAC does not invariably diminish with higher exhaust gas energy, as the EGTAC has constraints in recuperating exhaust gas, leading to a decline in fuel cell system efficiency after it reaches the peak.

(2) This paper proposes a method to estimate the exhaust characteristics of fuel cell systems by estimating the pressure of the cathode outlet gas of the stack using a state observer and calculating the mass flow rate based on it. The simulation results of 500s under three different load powers show that the maximum error rates of pressure and mass flow rate estimation are less than 13.84% and 12.1%, respectively, and the estimation error rates are less than 10% for 80% of the simulation time. This method achieves accurate tracking of the characteristics of the cathode outlet gas.

(3) The sensorless control technique presented in this paper aims to optimize the efficiency of the EGTAC and fuel cell system. It possesses excellent decision making capabilities and strong learning ability. The study demonstrates that the exhaust gas recovery rate of the EGTAC increased by 19.1% and 12.7%, while the fuel cell system's efficiency improved by 3.7% and 2.3% under high and medium loads, respectively, and the maximum difference is less than 6.5% compared with the control effect with sensors, which indicates that the suggested observer-based control approach for recovering energy from fuel cell exhaust can estimate gas characteristics accurately at the cathode outlet, offering guidance for controlling the exhaust gas recovery process. Consequently, the stability of the EGTAC operation is ensured, and the fuel cell system can achieve efficient output performance.

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